NUCLEAR SYSTEMS I (3rd Printing): THERMAL HYDRAULIC FUNDAMENTALS Neil E. Todreas and Mujid S. Kazimi

ERRATA

| PAGE (Line, Fig., Eq., Ex.) | ORIGINAL | CORRECTED |
|--|---|--|
| 31 (Table 2-3) | BWR, Reactor vessel, Linear heat rate, Core average (kW/m) is given as 19.0 | Should be 20.26 |
| 116 (Table 4-10, Cylindrical, last line | $\dots = \dots \left[\dots + \frac{1}{r^2} \frac{\partial v_z}{\partial \theta^2} + \dots \right] \dots$ | $\dots = \dots \left[\dots + \frac{1}{r^2} \frac{\partial^2 \upsilon_z}{\partial \theta^2} + \dots \right] \dots$ |
| 118 (Table 4-12, Spherical) | $p\frac{Dh}{Dt} = \dots$ | $\rho \frac{\mathrm{Dh}}{\mathrm{Dt}} = \dots$ |
| 131 (A.2) | where $\tilde{\alpha}_v = a$ local time-averaged void fraction. | where $\tilde{\alpha}_k$ = a local time-averaged phase fraction. Again for the gaseous phase $\tilde{\alpha}_k = \tilde{\alpha}_v$. |
| 133 (Line 1, above Eq. 5-21) | $\langle \mathbf{c} \rangle' = 0$ | $\overline{\langle \mathbf{c} \rangle'} = 0$ |
| 142 (bottom, title) | H Summary of Useful | I Summary of Useful |
| 232 (Line 15) | $\dot{Q}_{R} = \dot{m}c_{p}(T_{3} - T_{2}) + \dots$ | $\dot{Q}_{R} = \dot{m}c_{p}(T_{3} - T_{2'}) + \dots$ |
| 358 (Table 9-1) | where = $\frac{\text{small cross - sectional area}}{\text{large cross - sectional area}}$ | where $\beta = \frac{\text{small cross - sectional area}}{\text{large cross - sectional area}}$ |
| 378 (1 st paragraph) | (where $\varepsilon_{\rm M} = \dots$ | (where $\varepsilon_{M} = 0$), an intermediate or buffer sublayer, and a fully turbulent sublayer (where $\mu \ll \rho \varepsilon_{M}$) |
| 378 (Eq. 9-77) | $V_m = 0.187 V_{f\underline{L}}$ | $V_m = 0.817 V_{\text{G}}$ |
| 383 (Fig. 9-22, caption) | for parallel flow in a rod bundle. | for parallel flow in an infinite triangular array of rods. |
| 387 (Line 18) | (where H = axial lead of the wire wrap). | (where $H =$ the lead length or axial pitch of the wire wrap). |

| PAGE (Line, Fig., Eq., Ex.) | ORIGINAL | CORRECTED |
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| 389 (Eq. 9-93a) | $C_{\rm fT} = [0.8063 - 0.9022(\log_{10}({\rm H/D})] + 0.3256[\log_{10}({\rm H/D})]^2({\rm P/D})^{9.7}({\rm H/D})]^2$ | 1.79 2.0(P/D) |
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| | $C_{fT} = \left[0.8063 - 0.9022 \right]$ | $(\log_{10}(H/D)) + 0.3526[\log_{10}(H/D)]^2$ |
| | $(P/D)^{9.7} (H/D)^{(1.7)}$ | 78–2.0(P/D)) |
| 400 (3 Viscous) | Kays' defintions of K_e and K_c need to be added. | Kays' definitions for K_e and K_c , which apply to Fig. 9-36 and are consistent with the general definition of K in Eq. 9-23: |
| | | $\Delta p_{\text{form}} = \left[\left(1 - \beta^2 \right) - K_e \right] \frac{\rho v_{\text{ref}}^2}{2}$ |
| | | $\Delta p_{\text{form}} = \left[\left(1 - \beta^2 \right) + K_c \right] \frac{\rho v_{\text{ref}}^2}{2}$ |
| 402 (B Assumption) | Same as page 400 above. | Same as page 400 above. |
| 407 (Prob. 9-1) | Emptying of a liquid task | Emptying of a liquid tank |
| 409 (Prob. 9-5, Answers) | $\Delta p_{\text{entrance}} + \Delta P_{\text{exit}} = 11.73 \text{ kPa}$ | $\Delta p_{entrance} + \Delta p_{exit} = 10.73 kPa$ |
| | $\Delta p_{\rm T} = 94.59 \text{ kPa}$ | $\Delta p_{\rm T} = 93.59 \rm kPa$ |
| 427 (Table 10-6) | Table 10-6 | Table 10-4 |
| 440 (Eq. 19-80b) | $k_t^2 = \frac{1}{2} \left(\overline{\upsilon}_i' \overline{\upsilon}_i' \right) = \dots$ | $k_t^2 = \frac{1}{2} \left(\overline{\upsilon'_i \upsilon'_i} \right) = \dots$ |
| 527 (Fig. 12-3) | $q_{cr}'' = C_{1\rho g} h_{fg} [\dots]^{1/4}$ | $q_{cr}'' = C_1 \rho_g h_{fg} []^{1/4}$ |
| 529 (Fig. 12-4) | The boundary between regions B and C is not at the same axial elevation as the boundary between the regions of "Subcooled boiling" and "Saturated nucleate boiling". | |
| 529 (Fig. 12-4) | The boundary between regions D and E is not at the same axial elevation as the boundary between the regions of "Saturated nucleate boiling" and "Forced convective heat transfer through liquid film". | These two boundaries should be pictured at the same axial elevation along the heat transfer region. |
| 530 (Fig. 12-6) | Abscissa unlabeled | Abscissa is: quality, x |
| 535 (Eq. 12-19) | ST = | St = |

| PAGE (Line, Fig., Eq., Ex.) | ORIGINAL | CORRECTED |
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| 536 (Fig. 12-10) | $St = q''_w Gc_{pL} \Delta T_D$ | $St = q''_w / Gc_{pL} \Delta T_D$ |
| 554 (Eq. 12-49) | $h = \frac{k_g}{z} [\dots]^{1/4}$ | $h = C \frac{k_g}{z} [\dots]^{1/4}$ |
| 565 (Eq. 12-73a) | $x_{cr} = \frac{D_h}{D_e}(\dots)$ | $\mathbf{x}_{\rm cr} = \frac{\mathbf{D}_{\rm e}}{\mathbf{D}_{\rm h}} (\dots)$ |
| 574 (Problem 12-2) | $q_s'' = \dots \left[\frac{c_{p,\ell} (T_w - T_{sat})}{C_{s,f} h_{fg} P r_\ell^s} \right]$ | $q_{s}'' = \dots \left[\frac{c_{p,\ell} (T_{w} - T_{sat})}{C_{s,f} h_{fg} Pr_{\ell}^{s}} \right]^{3}$ |
| 577 (Eq. 5-72) | $\mathbf{h}_{\mathrm{m}} = \left\{ \alpha \rho_{\mathrm{v}} \mathbf{h}_{\mathrm{v}} = \dots \right.$ | $\mathbf{h}_{\mathrm{m}} = \left\{ \alpha \rho_{\mathrm{v}} \mathbf{h}_{\mathrm{v}} + \dots \right.$ |
| 656 (Table E-2, title) | of dry saturated steam pressure | of steam and water at saturation |
| 658 (Table E-3, title) | of dry saturated steam temperature | of steam and water at saturation |
| 695 (Index) | Froude number, 677 | Froude number, 675 |