#### **ASPECTS** OF **A** SEISMIC **STUDY** OF THE

MITR-

**by**

#### **GEORGE C. ALLEN,** JR.

#### Submi.tted in Partial Fulfillment **of** the Requirements for the Degree **of** Master **of** Science at the Massachusetts Institute of Technology

May, **1971**

Signature of Author Departmeht **of** Nuclear Engineering May 14, **1971** Certified **by\_** Thesis Sypervisor Accepted **by** Chairman, Departmental Committee on Graduate Students **Archives As. IST.Trc7 E JUN 15 1971**

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#### **ASPECTS** *OF* **A** SEISMIC **STUDY** OF **THE** MITR

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Submitted to the Department of Nuclear Engineering on May 14, **1971,** in partial fulfillment of the requirements for the Degree of Master of Science.

#### ABSTRACT

The design version of the Massachusetts Institute of Technology reasearch reactor (MITR-II) was analyzed subject to earthquake forces. The problem was divided into three major areas.

First, the reactor core tank and support structure were studied. The reactor can be adequately cooled and shutdown The reactor can be adequately cooled and shutdown if the core tank remains undamaged. Using a SABOR-5 computer program, the peak accelerations required to cause yielding of the core tank were calculated to be well above potential earthquake accelerations.

Second, the possibilities of potential damage to miscellaneous reactor systems were studied, The miscellaneous systems were studied to see if earthquake accelerations, resonance response, or differential motions would result in damage leading to major radioactive releases. No major potential hazards were discovered.

Third, the possibility of earthquake damage to the reactor stack was studied. An approximate analysis of the stack subject to dynamic earthquake shear and a 100 mile per hour wind was made. A case of a fallen stack was modeled to determine its efect on the containment building. The condetermine its efect on the containment building. servative calculations indicate that it is unlikely that the stack will fall and even if it were to fall onto the containment shell, it would not cause damage to the reactor core tank.

Within the scope of this report, it appears that the design MITR-II is adequate to provide required protection even in the event of the maximum expected earthquake motions.

Thesis Supervisor: David **D.** Lanning Professor of Nuclear Engineering

#### **ACKNOWLEDGEMENTS**

The author wishes to express sincere appreciation for the assistance of his thesis supervisor, Professor David D. Lanning. He would also like to thank James Kotanchik who made available the results of several SABOR-5 computer code calculations. The author also thanks Edward Barnett and the reactor operations staff for their assistance.

The author would also like to acknowledge the valuable assistance of Professor **J.** M. Biggs and Professor R. **J.** Hansen of the Department of Civil Engineering, and the advice of Professor **E. A.** Witmer and Barbara Berg of the Department of Aeronautics and Astronautics. He is also grateful to Chris Ryan for loan of a Schaevitz accelerometer. Special thanks go to Miss Schmidt for her help in preparation of this document.

Many of the calculations in this thesis were made using the System 360/M65 of the M.I.T. Computation Center.

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# LIST OF FIGURES





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# LIST OF **TABLES**



### Chapter **1**

#### SEISMOLOGY **AND** METHOD OF ANALYSIS

#### **1.1** INTRODUCTION

Earthquakes must be considered in the design of nuclear reactors, even in the New England area. Most earthquakes in New England pass without being noticed, for there are no less than several thousand minor earthquakes each year. The Massachusetts Institute of Technology research reactor (MITR-II) located in Cambridge, Massachusetts must be able to withstand earthquake motion without endangering the local populace. This work is an evaluation of major aspects of a seismic study of MITR-II.

The remainder of this chapter will cover the history of earthquakes in the Boston area and the seismic probability of the site area. The last section of this chapter will explain the seismic analysis sequence of MITR-II.

#### 1.2 HISTORY OF **EARTHQUAKES**

The Cambridge area lies in the Boston Basin which has been relatively free of earthquakes in recorded times.(Sl). The United States Department of Commerce in Earthquake History of the United States (H2) has the following to say about Massachusetts.

"In addition to feeling some of the more severe Canadian earthquakes and the New York and Grand Banks earthquakes of **1929, 17** (of intensity **5** and over on the Rossi-Forel scale) are listed for this state. In colonial times there were a large number of earthquakes in the northeast part of the state near Newburyport, and several of these, especially that of **1727 (75,000** square miles), were widely felt. That of **1755,** near Boston, was felt over an area of 300,000 square miles. The shock of **1925** in the vicinity of Boston was strong. Numerous moderate shocks have been felt in the southeast part of the state."

Massachusetts earthquakes of Rossi-Forel intensity of seven or greater in Earthquake Damage and Earthquake Insurance **by** John R. Freeman (Fl) as follows:



An explanation of the Rossi-Forel scale and a correlation with the Modified-Mercalli intensity scale is found in Figure 1.1. Historical accounts of Boston area earthquakes during **1727-1755** included such phrases as (Hl):

"...many chimneys were leveled with the roofs of the houses, and many more shattered and thrown down in parts..."

**"...the** gable ends of some brick buildings (were) thrown down and others cracked..."

"...(strong motions) continued about two minutes..."

#### **1.3 EARTHQUAKE ZONE AND** RETURN PERIOD

At the present time there is no standard Seismic Risk or Probability Map available on the United States that an engineer is required to follow. Such maps do however give a feel for the potential damage or expected maximum intensity in a given area. Examples of three Seismic Risk maps are shown in Figures 1.2, **1.3,** and 1.4. From these maps, the Cambridge MITR-II site appears to be in a region of moderate potential earthquake damage and have a maximum earthquake intensity of about **8** on the Modified-Mercalli scale. The relationship between Modified-Mercalli intensity and acceleration is shown in Figure *1.5.*

**<sup>A</sup>**different representation of earthquake activity is shown **by** the use of return periods (approximate frequency) of various accelerations in Figures **1.6** and **1.7.** The predicted return period for a **0.1 g.** earthquake (equivalent to VII on the Modified-Mercalli scale) for a Cambridge site would be approximately **1,000** years according to the maps of Milne and Davenport **(1969).**

#### 1.4 **LOCAL** SOIL CONDITIONS **AT** THE MITR-II SITE

The average soil conditions at the site are about **11** feet of miscellaneous fill overlying from **5** to **10** feet of soft organic silt and peat. Immediately below are approximately **10** feet of hard, medium to fine sand and gravel lying above more than **100** feet of Boston blue clay. The

reactor building and adjacent stack, as shown in Figure **1.8,** are on reinforced concrete mats founded on the hard sand and gravel **(Al).**

In general, earthquake motions are amplified and otherwise modified **by** near-surface geological features. At other locations, in a study **by** Tamura (Hl) it was found that the peak acceleration at ground surface was about twice the peak acceleration at a depth **of 300** meters (the same trend of increased motion from depth up to the surface occurred for both a deposit of soil and rock). Based on the assumption that most ways of estimating ground motions really apply for a very dense hard alluvium or for soft rock, Newmark and Hall suggested site factors to modify earthquake motions to make them apply for very soft ground or hard rock (Hl).

Newmark's Site Factors



The MITR-II is located on soft ground. Assuming a reasonable design peak acceleration of **0.15 g.** based on the area history and on the predicted return periods (Boston Edison's Plymouth Nuclear Power Station used approximately **0.15 g.** as its design peak acceleration) and applying Newmark's site factor of **1.5,** the estimated design peak earthquake acceleration at the MITR-II site would be **0.225** g.

#### **1.5** ANALYSIS OF THE MITR-II

The reactor shield is an integral unit with the remainder of the building. The building rests on a **<sup>70</sup>** foot diameter, **3** feet thick, heavily reinforced concrete pad. It is expected that the shield and pad will shift as a unit rather than cracking under seismic shock. On a more quantitative basis, a careful review of the seismic effects on the MITR-II has been made with the assistance of Professor Robert **J.** Hansen and Professor John M. Biggs from the Civil Engineering Department of MIT. Based on their experience with seismic effects, (Hl) it was concluded that the support for the main core tank of the MITR-II will not lose its structural integrity and hence will always be able to support the core tank. As shown in the MITR-II Safety Analysis Report **(Sl),** the reactor can be shut down **by** insertion of independently acting shim blades or **by** backup shutdown action of dumping the **D 0** reflector. It has also been shown in the 2 MITR-II Safety Analysis Report that the core will be adequately protected as long as H **0** remains in the tank for 2 natural convective cooling after the controls actuate to shut the reactor down. One problem which is particularly severe in regions of high seismic activity in the western United States but can be ignored for the MITR-II site, is the possibility of fault displacement through the site (Hl),

Chapter II of the following report, contains a detailed

analysis to determine the earthquake forces that would be required to cause a yield stress in the core tank itself. The core tank is analyzed for the operating case with the **D2 0** reflector dumped, being subject to various accelerations.

Chapter III discusses a study of seismic effects on several reactor systems such as control rods, miscellaneous piping, building penetrations, and floor loadings.

Chapter IV is seismic study of the brick stack adjacent to the reactor building.

Chapter V is a summary and list of potential recommendations.

# **FIGU** RE **I.**

# **RELATIONS** BETWEEN INTENSITY **SCALES** AND **ACCELERATION**



 $\bar{z}$ 



 $\sim$ 



 $(H<sub>1</sub>)$ 

H

 $\mathcal{L}_{\mathbf{Z}}$  ,  $\mathcal{L}_{\mathbf{Z}}$  ,  $\mathcal{L}_{\mathbf{Z}}$ 

**FIG URE** I.3



**SEISMIC** PROBABILITY MAP **OF** THE **UNITED STATES** II (H **)**

F**IGURE** 1.4



SEISMIC PROBABILITY MAP OF THE UNITED STATES-III (HI)



r. **6 FIG U** R **E**



**CONTOUR** MAP OF ACCELERATIONS **AS A** PERCENT OF OF **g** WITH **A 100** YEAR RETURN PERIOD FOR **EASTERN CANADA** (from Milne and Davenport, **1969)**

 $\overline{a}$ 



**CONTOUR** MAP OF.RETURN PERIOD IN YEARS FOR **ACCELERATION 'OF 0.1 g** IN **EASTERN CANADA** (from Milne and Davenport, **1969)**



 $\mathcal{C}$ 

#### Chapter 2

#### ANALYSIS OF THE MITR-II **CORE TANK**

#### 2.1 INTRODUCTION

The critical component of the MITR-II subject to seismic effects is the reactor vessel. Because **of** the extremely rigid structure supporting the reactor vessel, earthquake motion of the reactor vessel supports will be similar to soil motions at the foundation mat of the reactor building. The reactor can be maintained in a safe condition and the fuel adequately cooled if the core tank is not damaged and remains filled with water. Considering the reactor core tank as the critical component to be maintained, independent of the need of the outer containment building; failure of the stack or mechanical support facilities from earthquake motion can be tolerated, even in the unlikely case of the stack falling and hitting the outer containment building.

#### 2.2 SUMMARY OF **LOADS**

The configuration of the reactor core tank is shown in Figure 2.1 and the configuration of the inner vessel (flow shroud) is shown in Figure 2.2. Studies have been made of both the H<sub>2</sub>0 outer core tank and the inner vessel, including the flow shroud, to determine stress levels in each structure in terms of several loading parameters.

The complete set of loads consists of a gravity load and inertia loads that might be associated with both vertical

# F **I G U** RE 2.1

 $\sim 100$ 

OUTER CORE **TANK** , **ELEMENT DIVISIONS** FOR SABOR-5 **CALCULATIONS**



 $\mathcal{L}(\mathcal{L}(\mathcal{L}))$  and  $\mathcal{L}(\mathcal{L}(\mathcal{L}))$  . The contribution of

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**FIG U RE** 2.2

**INNER VESSEL AND** FLOW **SHROUD ELEMENT DIVISIONS** FOR S A B OR-5 **CALCULATIONS**



 $\mathcal{L}_{\mathcal{C}}$ 

 $\mathcal{L}^{\text{max}}_{\text{max}}$  , where  $\mathcal{L}^{\text{max}}_{\text{max}}$ 

and horizontal seismic motion of the structure. For all cases, the vertical acceleration was set at **2/3** of the horizontal level, following the suggestion of Professor Hansen of the MIT Civil Engineering Department. The discussion of the loads given below will be divided into two sections: **(1)** loads for the inner vessel; and (2) loads for the outer core tank.

#### 2.2.1 Inner Vessel Loads

The loads associated with the inner vessel and flow shroud must be considered in terms of its geometry. The fuel element hexagonal container is porous and hence no water inertial loads are acting upon it. The support ring of the fuel element container will be subject to water and structural inertial loads only, but no hydrostatic loads since both the inner vessel and outer core tank are connected. Loads on the vertical section of the flow shroud are due to the inertia of the metal itself and the inertial loads due to the contained water. These inertial loads can be characterized **by** the expression (for the horizontal component).

$$
F = \mathbf{P_w} R_i \cos \mathbf{\Theta} - 90^\circ \le \mathbf{\Theta} \le 90^\circ
$$
  
= 0 90°  $\le \mathbf{\Theta} \le 270^\circ$  (2.1)

where  $\mathcal{C}_w$  = density of water

- Ri **=** inside radius of the inner vessel
- $\Theta$  = angle measured from the direction of horizontal motion. (The orientation of is shown in Figure **2.3).**

The above set of loads is conveniently represented using

**-27**

4 terms of a Fourier cosine series which yields the expression:

$$
F = 0.31831 \tP_wR_i + 0.5 \tP_wR_i \cos\theta + 0.21221 \tP_wR_i \cos 2\theta - 0.04244 \tP_wR_i \cos 4\theta
$$
\n(2.2)

This form returns **99%** of the peak load and is an adequate representation of the load for this study.

The effect of the various loads have been calculated **by** using a computer program SABOR-5 (K2).

For convenience in running the SABOR-5 program, a small program was written to generate a set of loads as the input for SABOR-5. **A** listing of this program is included in Appendix B.

When SABOR-5 is run for the inner vessel, the program considers the nodes at which the inner vessel is supported **by** the outer core tank to be restrained, and it calculates a set of forces to be applied to the outer vessel at the corresponding outer vessel nodes.

The effect of the fuel elements and their supporting material is included **by** considering that portion of the structure as a lumped mass at its center of gravity, and equivalent ring loads are calculated and applied at the edge of the support flange.

#### 2.2.2 Outer Core Tank Loads

Because of the narrow clearance between the flow shroud and the core tank, water inertial loads resulting from sideways motion are neglected. In the lower portion of the

vessel, the contribution of the water inertial load is small, (due to the small local radius and the large portion occupied **by** the core) and can be neglected in comparison to the loads due to the very large contribution from the upward motion and the hydrostatic head.

For the core tank, a small program was again written to provide the input to **SABOR-5. A** listing of this program is included in Appendix **C.**

#### **2.2.3** Calculational Model

The reactor vessel is composed of two major components, the outer core tank and the inner flow shroud. In the calculational model, the outer core tank was divided into 94 finite elements and the inner flow shroud was divided into **79** finite elements. The stress on each of these elements was determined **by** using the SABOR-5 computer program. These elements are shown in Figures 2.1 and 2.2.

Several problems were encountered in the modeling and in the calculation of loads. Primary among these is the problem associated with the support geometry between the inner vessel and the outer core tank. The physical support consists of 12 feet equally spaced circumferentially between the inner and outer vessel. This construction introduces physical asymmetry into the geometry and since SABOR-5 handles asymmetric geometries only with great difficulty, some study of the modeling of the structure in that area was made.

SABOR-5 models all geometries as axisymmetric structures. Local asymmetries are modeled **by** "smearing" the structure in that local area. For structural elements such as the twelve supporting feet, the SABOR-5 program would generate results for a continuous ring between the inner and outer vessels (equivalent to increasing the metal density **by** appropriate amounts for the elements in the feet area). But in this model, serious errors can result in the calculated local stress distribution in the area where the feet rest upon the outer core tank, and for this reason a detailed study of this problem was undertaken.

For this detailed study, the SABOR-5 calculated loads at the feet of the inner vessel were lumped into values at each of the twelve feet and a higher order Fourier series loading for the outer core tank was computed from them. Harmonics **0, 1, 11,** 12, **13, 23,** 24, 25, **35, 36** were used. This representation of the load at the feet was within **5%** of the exact value, and the load midway between the feet was negligible. The local stress distribution in the region of feet on the outer vessel was then computed. The values obtained were compared with those of the continuous mode. This comparison yielded the result that the peak stresses were nearly **50%** greater than those obtained from a continuous model of the support loads.

The two maximum stress locations on the outer tank were

located near the bottom center of the tank and in the feet area at the element above the feet. Thus to be conservative, the feet area stresses reported in AppendixAare the maximum stresses in the feet area calculated using the continuous case and increased **by** a factor of **60%.**

**2.3** DESCRIPTION OF THE SABOR-5 COMPUTER PROGRAM

The analysis **by** using the SABOR-5 Computer Program treats linear-elastic, static, load-deflection behavior of meridionally-curved, variable thickness double **-** and/or single-layer, branched thin shells of revolution which may be subjected to concentrated or distributed external mechanical and/or thermal loads. In the present analysis, it is assumed that the structure is axisymmetric in terms of both geometry and material properties; hence, when the structural deformations are expressed as sums of Fourier harmonics of the circumferential coordinate  $\Theta$ , the equilibrium equation for the structure consists of a set of harmonically-uncoupled load-displacement equations (ie., there is a separate set of equations for each harmonic of the structural response).(Kl). The harmonic deflection coefficient may be determined **by** solving these equations for each significant loading harmonic present, and may then be summed to obtain the complete deformation. The requirement of material axisymmetry means that while nonuniform and/or asymmetric temperature distributions may be treated,

the material properties,  $E, \nu, \infty$ , must be independent of the local temperature, but more precisely independent of location

#### 2.4 OUTLINE OF THE ANALYSIS

The application of the discrete-element procedure may be divided into three phases: structural idealization, evaluation of element properties, and analysis of the complete structure. In the analysis of both the inner and outer vessels, the structures were modeled as single-layer elements.

In the discrete-element formulation, the actual structure is replaced **by** an assemblage of geometrically-compatible discrete elements. As previously stated, for the present analysis, the basic discrete elements to be employed are single-layer. For a single-layer element only four quantities are necessary to fully describe the state of deformation within an element. These quantities for each element are:

(a) Midsurface meridional displacement,  $\mu$ 

**(b)** Midsurface circumferential displacement, V

(c) Normal displacement, W

(d) Total meridional rotation,  $\frac{\partial w}{\partial x} + \mu \frac{\partial w}{\partial y}$ as **CAs**

In the above expressions  $\phi$  is the meridional slope of the element. One may reduce the number of degrees of freedom necessary to describe the deformation state of each element **by** choosing a set of generalized displacements corresponding

to the nodal values of the displacements in each element. Then the choice of a reasonable assumed (analytic) function for the interior displacements, which also includes interelement displacement compatibility, provides a complete representation of the overall deformation state of the element.

At the bounding nodes of the  $\texttt{P}^{\texttt{th}}$  element (say nodes **q** and **r**) let generalized forces  $Q_1$ , **q** ...  $Q_n$ , **q**,  $Q_1$ , **r** ...  $Q_n$ , r (where n=4 for single-layer elements) be defined corresponding to the generalized displacements at the node. The application of the Principle of Stationary Total Potential Energy yiblds the equations for static equilibrium for the Pth element; as a consequence **df** the condition of structural axisymmetry, these force-displacement relations are harmonically uncoupled. In matrix form, the force-displacement relation for the **jth** harmonic A-series Fourier displacement component becomes:

 $\left[ \begin{array}{c} K_{p}(j) \end{array} \right]$   $\left\{ \begin{array}{c} q_{p}(j) \\ q_{p}(j) \end{array} \right\}$   $= \left\{ \begin{array}{c} Q_{p}(j) \end{array} \right\}$ where  $\left[ K_{p}(\mathbf{j})\right]$  is the element stiffness matrix. Imposition of nodal compatibility at interior nodes of the complete structure requires that at an interior node r, bounded **by** elements **p** and **q** the following relationship must be satisfied:

$$
\begin{Bmatrix}\nq_1 \cdot r \\
q_2 \cdot r \\
\vdots \\
q_n \cdot r\n\end{Bmatrix} = \begin{Bmatrix}\nq_1 \cdot r \\
q_2 \cdot r \\
\vdots \\
q_n \cdot r\n\end{Bmatrix}
$$
\n(2.3)\n
$$
\begin{Bmatrix}\nq_1 \cdot r \\
q_2 \cdot r \\
\vdots \\
q_n \cdot r\n\end{Bmatrix}
$$

The total potential energy  $(\pi_{p})$  for the n discrete elements, including the strain energy as well as the potential energy of all of the virtual-work-equivalent nodal loads (for both distributed and ring loads), and the imposition of nodal compatibility then becomes expressed in terms of the **N** independent nodal displacement of the complete assembled discretized structure. The equilibrium equations for the entire structure are then obtained **by** setting  $\delta \pi_{p}$  = 0, where only displacement variations are permitted. For the  $j<sup>th</sup>$  harmonic A-series Fourier displacement harmonic, for example, these read:

$$
\left[\begin{array}{c}\nK(j) \\
K(k)\n\end{array}\right] \left\{\n\begin{array}{c}\nq(j) \\
R(k)\n\end{array}\n\right\} = \left\{\n\begin{array}{c}\nF(kj) \\
K(k)\n\end{array}\n\right\} \n(2.4)
$$

In the above equation, **N** is the total number of degrees of freedom associated with the complete structure, and the ith term  $F_i$ <sup>(j)</sup> of the generalized force vector is the sum of all the **jth** harmonic generalized forces from all the individual discrete elements and from the **jth** harmonic generalized ring-type loads, both of which are associated with ith degree of freedom of the complete assembled discretized structure. Also,  $K(j)$  represents the (assembled) stiffness matrix for the complete structure.

For most practical applications, the physical structure will be restrained in some fashion such that one or more generalized displacements will be known before the general solution is obtained. **A** solution for the complete

displacement field then will not be found from Equation (2.4) but from a reduced equation from which the restrained degrees of freedom have been deleted. The reduced equation is similar in form to Equation (2.4) and follows:

 $\left[ \begin{array}{cc} K^{(j)} \end{array} \right]$   $\left\{ \begin{array}{cc} q^{(j)} \end{array} \right\}$   $= \left\{ \begin{array}{cc} F^{(j)} \end{array} \right\}$  (2.5) (N-R) x (N-R) **(N-R)** x **1** (N-R) x **1** where R is the number of known or prescribed generalized

displacements.

Equation **(2.5)** may be solved for the unknown-generalized displacements using any appropriate method. **A** similar equation may be written for each loading harmonic present. The total generalized displacement may then be found **by** summing the contributions due to each loading harmonic. Since all of the **N** displacements are now known, they may be used to determine other information such as:

- (a) strains **by** use of the appropriate strain displacement relations and
- **(b)** stresses and/or stress resultants.

The detailed loads programs required to carry out the above **-** outlined discrete-element analysis for the specific case of the MITR-II reactor core tank and inner flow shroud are included in Appendices B and **C.** There are no thermal loadings in our analysis.

### 2.5 **RESULTS** OF THE **CALCULATIONS**

SABOR-5 gave results at the midplane, inner and outer

surfaces of each element of the vessels being analyzed for various circumferential theta stations. Comparison of the computer results with hand calculations showed agreement between the two methods (the computer method appears to be the more conservative). This comparison was made for the case of zero acceleration, static operating conditions (Table **A.1).** SABOR-5 also showed agreement between the static case and freebody stress analysis (Table **A.2).** On the basis of this agreement and from previous experience with the SABOR-5 program as documented in the work **by** Witmer and Kotanchik, (Wl) the program is taken to give valid estimates of the stresses.

Extremely small stresses were calculated to occur in the inner flow shroud (Table **A.3),** in comparison with the outer core tank. The outer core tank is therefore shown to be the critical part.

The outer core tank has been analyzed for the following four cases:

- (a) static operating case **(STC)**
- **(b) STC + 0.5 g.** horizontal **+ 0.33** g. vertical
- (c) **STC + 1.5 g.** horizontal **+ 1.0 g.** vertical
- (d) STC  $+ 4.5$  g. horizontal  $+ 3.9$  g. vertical

The peak stresses occurred on the inside of station **50** (Figure 2.1). This peak stress is actually the peak stress as given **by** the continuous case increased **by** a factor of 60%. The summary of the calculated results are in Table  $A.4$ .




 $\frac{8}{8}$ 

#### Table **2.1**

## LIMITING ACCELERATIONS **ON** CORE **TANK**



 $\bar{1}$ 

 $\mathcal{L}_{\rm{in}}$ 

The analysis showed that stress increased linearly with acceleration (Figure 2.4). Thus, **by** extrapolation of the results to the condition for which the peak stress equals the working stress, and the case where the peak stress equals the yield stress, the limiting accelerations were derived as shown in Table 2.1.

## **2.6** DETERMINATION OF THE **FUNDAMENTAL FREQUENCY** OF THE WATER FILLED CORE **TANK**

The fundamental mode of the water-filled core tank is determined **by** a numerical iterative method commonly known as the Stodola and Vianello method (H3). For a structural system the following equation holds.

 $\overline{K} \ \overline{U} = w^2 \ \overline{M} \ \overline{U}$  (2.6)

where



Rearranging terms

$$
\frac{1}{w^2} \quad \overline{U} = \overline{K}^{-1} \overline{M} \overline{U} \tag{2.7}
$$

letting

$$
\lambda = \frac{1}{w^2} \quad \text{and } \overline{k}^{-1} \ \overline{M} = \overline{a} \quad (2.8)
$$

then the Stodola and Vianello method can be used to solve

an equation of the form:

$$
\bar{a} \bar{x} = \lambda \bar{x} \qquad (2.9)
$$

where after assuming an initial displacement, the process is iterated until  $\lambda$  converges on a maximum and thus obtaining the smallest natural frequency. The mass matrix and stiffness matrix are obtained **by** running the SABOR-5 program for the outer core tank. **A** computer program was written to convert the SABOR-5 mass and stiffness matrix to the X-Y-Z coordinate system and to increase the mass matrix **by** appropriate lumped masses corresponding to the water in the tank. The program also performs the inversion of the stiffness matrix and performs the iteration process for an inputed assumed original normalized displacement (a listing of this computer program is included in Appendix **D.)** After **101** iterations the solution had converged on:

> $24.8$  cycles/sec. = 1st mode of water filled core tank

Thus resonant amplification does not appear to be a problem. (The initial assumed displacements are shown in Figure **D.l)**

#### **2.7** ANALYSIS OF **RESULTS**

**A** detailed analysis has been made to determine the earthquake forces that would be required to cause a yield stress in the core tank itself. Vector forces on the tank were considered to be largest in the horizontal direction and a force of **2/3** of the horizontal force was simultaneously applied downward (upward acceleration) in the vertical direction. The results of these calculations indicate that a combined horizontal acceleration **of 5.1 g.** and a vertical acceleration **of** 3.4 **g.** will cause a peak stress near the feet area of the core tank of **9500** psi which is equal to the yield stress limit for the aluminum tank.

It should be noted that these conclusions apply to the reactor structure and reactor core tank. It is conceivable that the effect of an earthquake could cause some damage to the reactor piping or building structure at lower accelerations; however, the action of the antisiphon valves would prevent a loss of the necessary  $H<sub>2</sub>0$  coolant in the main core tank.

**A** summary of conclusions reached on the seismic effects has been prepared **by** Professor Biggs of the MIT Department of Civil Engineering who states that:

"Based upon the seismic criteria commonly used for nuclear power plants, the Design Basis Earthquake for the Cambridge area would probably have a maximum acceleration of about 0.2 **g.** This estimate considers both the seismicity of the region and the fact that Cambridge is an area of relatively soft soil conditions.

"The structure supporting the research reactor is a massive, rigid concrete block extending from the bottom of the foundation to the point of reactor vessel support. Therefore there would be little, if any, amplifications of

the acceleration in the structure itself, i.e., the acceleration at the reactor support would be essentially the same as at the bottom of the foundation.

"However, since the support system is a mat foundation on relatively soft soil, a certain degree of soil-structure interaction is to be expected. This tends to increase the fundamental period of the structure and to make the motion of the foundation somewhat different than that occurring in the undisturbed soil.

"The soil-structure interaction in this case would be almost entirely swaying, or horizontal shearing, in the soil. This type of behavior involves very high damping. As a consequence, there would be little amplification of the ground acceleration,  $\dot{\mathbf{a}}\cdot\mathbf{e}\cdot\mathbf{n}$ , the maximum acceleration of the rigid foundation would be essentially the same as that predicted for the ground, or 0.2 **g.**

"The natural period of the reactor vessel is very short compared to that of the soil-structure foundation system. Therefore, there is no possibility of resonance between the vessel and the supporting structure.

**"All** of the above leads to the conclusion that the maximum response acceleration of the reactor could be only slightly greater than the maximum ground acceleration of 0.2 **g.**

"It has been computed that the reactor is capable of withstanding (at yield stresses) static forces corresponding

to *5.1* **g.** horizontal acceleration and simultaneously 3.4 **g.** vertically. It is not conceivable, even under the most unfavorable circumstances, that the response to earthquake motions would be more than a small fraction of these amounts."

#### Chapter **3**

#### **GENERAL AREAS** OF SEISMIC INTEREST IN MITR-II

#### **3.1** INTRODUCTION

Many miscellaneous areas of seismic concern exist in a nuclear facility. The following areas of the MITR-II will be covered in this report chapter:

1. Reactor Floor Design Loadings

2. Control Rods

**3.** Piping

4. Building Penetrations

**5.** Seismic Instrumentation

**6.** Temporary Shield Walls

No problems were discovered that would result in a potential reactor hazard for the design of the MITR-II.

#### **3.2** REACTOR FLOOR DESIGN LOADINGS

Referring to Figure **3.1,** a six foot ring around the reactor was designed for a live load **of 3,000** pounds per square foot; and the balance of the floor was designed for **2,000** pounds per square foot. The total design live load of the floor was 2,000 kips **( 1** kip **= 1,000** pounds) and the lattice facility area of the reactor floor was designed to be fully loaded (F2).

Figure **3.2,** shows a simplified representation of the MITR-I lattice facility and the proposed MITR-II lattice facility (the MITR-I lattice facility is decreased in height



# LOADINGS



**by** six feet). The maximum local loading and the approximate total load for both the MITR-I and the proposed lattice facility are shown in Table 3.1.

#### Table **3.1**

#### LATTICE FACILITY **LOADS**



While the loads due to both structures are well within the total design live loads, and the probability of other areas being fully loaded is very small, both lattice facilities yield local loads above the design 3,000 pound per square foot (psf) within six feet of the reactor and 2,000 psf beyond six feet from the reactor. During construction of the MITR-I lattice facility, careful measurements of the reactor floor were made to determine any deflections of the floor because of the lattice loading. No measurable deflection was found.

While the reactor floor has shown no signs of yielding or deflecting under the MITR-I lattice facility loading (which is not surprising because of the generous conservatism shown in designing the reactor building (F2)), it is

difficult to predict how much more additional loading the floor could safely take in that area, because it is already loaded at about twice its design value. However, for the proposed MITR-II lattice facility, it can obviously be stated that the floor area in the vicinity of the lattice facility will take a **25%** increase in load without damage, because the reactor floor had safely supported the MITR-I lattice facility (Load MITR-I Lattice **= ( 1 + 0.25 )** Load proposed MITR-II Lattice). The proposed lattice floor area will in effect, have been tested for a **0.25 g.** vertical acceleration **by** the experience with the MITR-I lattice. **A** vertical acceleration of **0.25** is greater than the peak potential vertical acceleration. Horizontal earthquake motions are resisted **by** steel bands around the lattice facility.

In any event, although failure of the reactor floor in the area of the lattice facility might cause damage to the primary system piping in the equipment room, there would be no damage to the core tank or the core tank supporting structure.

#### **3.3** PIPING

The piping systems in the MITR-II reactor have short period fundamental modes, well above the normal earthquake frequencies. The longest unrestrained run of a major pipe is the light water coolant pipe which runs from the equipment room to the core tank (the pipe is actually restrained

against large motions **by** the compactness of the area through which it passes). The first fundamental mode of this pipe is **97** cycles per second (Calculation is in Appendix **E).** Resonance response of the piping appears unlikely.

The need for flexibility in piping to accommodate thermal movement provides sufficient flexibility for differential movements of equipment during earthquake motions. It is recommended that consideration be given to lateral restraints of small piping in the following systems to assure that adequate seismic restraints are provided:

1. Ion Exchange Unit

2. Heavy Water Cleanup System

**3.** City Water Pipe

4. Helium supply system to D<sub>2</sub>0 gas holder

5. **D**<sub>2</sub>0 Sampling system

With the restraint of the above systems, the piping does not appear to be a major concern because of short runs, numerous restraints, and low pressures.

#### 3.4 SEISMIC **EFFECT ON** CONTROL RODS

#### 3.4.1 Description of Control Rods Assembly

The control rod assembly is shown in Figure **3.3.** The absorber blades travel in slots in the core housing with a nominal **1/16** inch clearance all around. The blade is offset, attached to a magnet armature rod that moves in a slit cylindrical guide tube. Analysis will be made of the increase in rod drop time from seismic motion, rod whip during earthquake motion, and the blade displacement from a **1g.** lateral acceleration.

#### 3.4.2 Drop Time

In the MITR-II, scram is accomplished **by** interruptingan electric current to the magnets **by** which the rods are suspended so that the rods are free to fall in their guide tubes. If these guide tubes can be considered frictionless, lateral forces will be unimportant (lateral forces will be considered in Section 3.4.3). Suppose a scram is initiated during an earthquake that is causing the entire reactor structure to vibrate in the vertical direction with a period on the order of **0.1** or 0.2 seconds and with acceleration varying accordingly  $(\frac{+}{\cdot}, 1, \varepsilon, \cdot)$ , which is typical of strong-motion earthquakes as recorded **by** vertical component seismometers) **(Nl).** When the current breaks, the control rod, along with the reactor, will have either upward, downward, or zero velocity with respect to the earth's mass as a whole. Since the magnitude of the vertical ground displacement in typical strong-motion quakes has rarely been known to exceed  $\frac{1}{2}$  2 centimeters, the effect of any change in total travel on roddrop time is insignificant. The effects of initial velocity of the rod at the time the magnet releases may be more significant. Suppose that at release the rod has an upward velocity of **0.3** ft/sec, **(Nl)** which is not unreasonable in



## FIGURE 3.4

## ROD IN CONTROL ROD GUIDE TUBE



strong quakes. The rod, once free, must continue upward until this velocity is reversed **by** gravity, which causes a theoretical delay of

$$
\frac{2v}{g} = \frac{2 (0.3)}{32.2} = 0.02 \text{ seconds}
$$
 (3.1)

Since the individual motions and reversals of the core and control rods imposed **by** earthquakes are erratic both in time and magnitude, a detailed analysis of all probably sequences of events in this initial split second is probably not meaningful, however, as a worst case assumption, one can assume a delay in the beginning of the free-fall drop cycle on the order of 0.02 seconds. **A** time delay of 0.02 seconds does not appreciably change the average drop time of **0.68** seconds.

#### *3.4.3* Rod Whip During an Earthquake

Consider the rod in the control rod guide tube as shown in Figure 3.4. Assume that the reactor is being accelerated to the left at a rate **g'** due to the earthquake, that the rod is rigid, and that its density does not vary along its length.

If the center of mass of the rod is within the guide tube, the effect of lateral acceleration will be to develop small friction forces between the rod and guide. Since the lateral acceleration g' will probably not exceed 10 ft/sec<sup>2</sup>  $(\sim\!1/3$  g) even in a very strong earthquake, these forces will generally be small, depending on the friction coefficient and the mass of the rod. For instance, in the MITR-II the rod

weighs approximately **25** lbs. Assuming a conservative friction coefficient of **1.0 (01),** the retarding friction force will be:

$$
\frac{25 \text{ lbs}}{32.2 \text{ ft/sec}^2} \times 10 \text{ ft/sec}^2 \times 1.0 = 7.8 \text{ lbs.}
$$
 (3.2)

This is not a constant retarding force. Actually, acceleration can reverse direction several times during the rod fall, varying from zero to  $\pm$  10 ft/sec<sup>2</sup> (assumed maximum, **(Nl));** thus the rod could rub alternately on opposite sides of the guide tube as it descends.

Considering Figure 3.4 again, a different situation arises if the center of mass of the rod is outside the guide tube. In this case the rod, with the greater fraction of its mass outside the guide tube, will pivot about Z, and resulting reactions at Z and **Q** can become large due to the lever action of the whipping rod. When the sum of these reactions  $(R_1$  and  $R_2$ ), multiplied by the coefficient of friction, exceeds the weight of the rod, it will not fall under the influence of gravity. For the MITR-II, the rods are keyed in the guide tube, thus a rotary motion may not develop so that this retarding effect is continuous for the duration of the earthquake.

**A** condition under which rod jamming could occur is simply derived as follows:

Referring to Figure 3.4:

For acceleration  $\ddot{x}$  to the left, the summation of

horizontal forces is

$$
R_1 = \frac{w\ddot{x}}{g} + R_2 \qquad \text{we right of rod} \qquad (3.3)
$$

The summation of moments about  $R_1$  yields

$$
R_2 a = \frac{w\ddot{x}}{g} \left(\frac{q}{2} - a\right)
$$
  
or 
$$
R_2 = w\ddot{x} \left(\frac{q}{2} - a\right)
$$
 (3.4)

Substituting Equation (3.4) into Equation **(3.3)** yields

$$
R_1 = \frac{w\ddot{x}\mathbf{Q}}{2ag} \tag{3.5}
$$

In order for the rod not to jam,

**g** 2a

 $\mu$ (R<sub>1</sub> + R<sub>2</sub>) must be less than w where  $\mu$  is the coefficient of friction.

Thus

$$
\mu_{\frac{w\ddot{x}}{g}}\left[\frac{\mathbf{q}}{2a} + \frac{\mathbf{q}}{2g} - 1\right] = \mu_{\frac{w\ddot{x}}{g}}\left[\frac{\mathbf{q}}{a} - 1\right] < w \qquad (3.6)
$$
\nor

\n
$$
\frac{\mathbf{q}}{a} < 1 + \mathbf{g}
$$
\nor

\n
$$
\left[1 + \mathbf{g}\right]^{-1} < \frac{\mathbf{a}}{1} \qquad (3.8)
$$

As an example let  $g = 32.2 \text{ ft/sec}^2$  $x = 10$  ft/sec<sup>2</sup>  $\mu = 1.0$ 

then

**1 <** a **(3.9)**  $4.22$   $9$ 0.238  $\lt \frac{a}{2}$  $(3.10)$ 

For the MITR-II, the minimum  $a = 23$   $1/8$  inches and

FIGURE 3.5

MODEL USED TO DETERMINE CONTROL BLADE DISPLACEMENT

 $\sim 10^6$ 

 $\sim$ 



## FOR 19 HORIZONTAL

 $\mathbf{Q}$  = 51 1/8 inches thus

 $a = 23 \frac{1}{8} = 0.45$  > 0.238 (3.11) **<sup>31</sup>51 1/8**

Thus it appears, that for the MITR-II, rod whip will not prevent the rods from dropping during an earthquake.

The actual control blades, themselves, cannot whip under earthquake motion because they are constrained at the bottom **by** their slots, and at the top **by** the control guide rod. The approximate displacement of the rod guide for a **lg** loading was calculated to determine if a large displacement of the control blade might occur which could result in a jammed blade (Calculation is found in Appendix F). The model is shown in Figure  $3.5$ . The displacement  $\Delta x$ , at the end of the blade, shown in Figure **3.5** for a **1 g** lateral load was found to be .00634 inches. This is a negligible displacement and according to Mr. Barnett (MITR-II design staff), this will have no effect on rod drop.

#### **3.5** BUILDING PENETRATIONS

Earthquake motion could conceivably cause differential motions between the reactor building and nearby buildings and ground. The reactor building is on a heavily reinforced concrete pad which will shift as a unit as a result of earthquake motion. The reactor building is separated from adjacent structures **by** a gap in the case of the stack structure and **by** a felt "seismic" separation in the cases of the entrance air

FIGURE 3.6

## SEISMIC SEPARATION OF AIR LOCK



locks and utilities building. Figure **3.6** shows a detail of the seismic separation around the personnel air-lock. The reactor building is able to move independently of the surrounding structures.

Rigid penetrations attached to the reactor building might be broken during an earthquake due to potential differential seismic motions. This problem is particularly acute for below grade penetrations because of lack of freedom of motion of buried pipes. A list of all reactor building penetrations is found in Table 3.2.

The spent fuel pool is entirely below ground water level and breach of the tank would cause leakage of ground water into the spent fuel. It thus appears unlikely that the spent fuel pool would become a radiation hazard before the tank could be repaired.

Special building penetrations for experimental facilities, such as the liquid helium production system and the pneumatic tube sample transfer system, are made in a manner to prevent any radioactivity release. These penetrations can be sealed **by** automatic isolation valves and **by** manual operational valves that can be closed from outside the reactor penetrations **(Sl).**

The emergency core spray is to be supplied **by** two redundant systems connected to city water, The connection to city water in the utilities room is to be **by** a flexible pipe

## Table **3.2**

 $\mathcal{L}^{\text{max}}_{\text{max}}$ 

 $\overline{a}$ 

#### **LIST** OF PENETRATIONS



 $\mathcal{A}^{\pm}$ 

 $\Delta$ 

#### Table **3.2**

(Continued)



Electric Service

 $\bar{\epsilon}$ 

 $\bar{\beta}$ 

No. of Penetrations



to allow for relative building motions.

If the pipes leading to the waste storage tanks (or the tanks themselves) are damaged **by** an earthquake, there is a potential leak of radioactive material into the groundwater. It is not intended that the waste storage tanks will be used for **highly** active waste **(Sl).** In the past twelve years, the sampling prior to discharge has shown that the solutions discharged from the waste tanks has not required extra in-tank dilution prior to discharge into the sewer system with final ocean discharge. Accidental release of this material into the ground water is not predicted to create an off-site concentration above permissible limits in occupied areas.

Although rupturing of any rigid reactor building penetration due to potential differential earthquake motions will not simultaneously cause a major release of activity, the broken penetrations might cause a possible breach in the reactor containment. If an internal  $D_2$ <sup>0</sup> pipe were to be broken at the same time as the breach in the containment, there would be a potential release of tritium **by** evaporation. Calculations have been made in the MITR-II Safety Analysis Report **(Sl)** which indicate that in the event of a rupture of both the  $D_2$ <sup>O</sup> system and the containment system, the offsite exposure to tritium activity would remain below permissible yearly averaged limits for at least two days. Thus, there would be ample time to evaluate the situation and take

#### appropriate action.

#### **3.6** SEISMIC INSTRUMENTATION

For a 14-day period from April **1, 1971** to April 14, **1971,** a Schaevitz **1 g** accelerometer was attached to the reactor building to measure expected everyday building accelerations. The accelerometer was attached to the reactor building shield wall nearest the reactor stack at a position about two feet above the reactor floor level. The electronics of the accelerometer setup used are shown in Figure **3.7** and the accelerometer was calibrated using the force of gravity. The accelerometer was oriented for five days in an approximate north-south direction (normal to the wall) and for five days in an east-west direction (parallel to the wall). For the remaining four days, the accelerometer was used to measure a vertical component of the acceleration.

The MITR-II site is located in an industrialized section of Cambridge, Massachusetts. The site is also adjacent to a railroad right of way. Numerous ground motions result from passing trucks and trains. The accelerometer measured a peak acceleration of these motions and not their frequency. The plot of peak accelerations was less erratic during weekends when the reactor was shut down.

The peak acceleration measured during the 14-day period occured when the accelerometer was aligned parallel to the shield wall and a train passed on the tracks adjacent to the



## ACCELEROMETER SIGNAL CIRCUIT



 $\mathcal{S}$ 



site. The accelerometer output for this occurance is shown in Figure **3.8.** This peak acceleration corresponds to about a **0.03 g** horizontal acceleration. The reactor operator also noted fluctuations in various galvanometer needles during the passing train.

There are numerous commercial strong motion accelerographs and seismic triggers available with actuating accelerations between **0.005** g to **0.05 g.** Because of the numerous industrially related ground motions at the MITR-II site, any proposed seismic trigger should actuate at between 0.04 **g** to **0.05 g** in order to minimize any false "seismic" alarms. Because of major interest in the safety of the core tank, an optimal location of any seismic instrumentation would be on the core tank support structure.

While seismic instrumentation would give the reactor operators the best analysis of building motions, the intensity of the following phenomena will also give the reactor operator a feeling for the extent of earthquake motions:

- 1. Fluctuation of galvanometer needles
- 2. Swaying of overhead lights
- **3.** Shaking of equipment

4. Movement of floor

#### **3.7** TEMPORARY SHIELD WALLS

Temporary shield walls of numerous unbonded concrete or lead bricks might fail during earthquake motion. Consequently

temporary shield walls should not be used in the MITR-II where their failure will result in an unacceptable offsite release of radioactivity or where the temporary shield failure could damage important reactor control mechanisms.

#### Chapter 4

#### REACTOR **STACK**

#### 4.1 INTRODUCTION

The **150'** reactor stack, which is adjacent to the reactor building, is a possible area of concern in the event of an earthquake. The stack is of the unlined brick variety. As shown **by** Figure 4.1 of the reactor site, assuming that an earthquake has an equal probability of occurring from any direction, the probability that the stack will fall into zone II, ie., hit the reactor containment, is about **.25.** (Note: this assumes the primary mode of failure shown in Figure  $4.1$  as being the worst case, since in higher modes of failure, no material would drop too far from the original vertical position)

#### 4.2 APPROXIMATE **EARTHQUAKE** ANALYSIS OF **STACK**

An approximate method of determining the dynamic earthquake shear at various horizontal sections of the stack is to consider the Recommended Lateral Force Requirements **(1959)** of the Structural Engineers Association of California **(SEAOC).** The procedure specified is based on only the first mode of the structure, which was assumed to be the failure mode of most concern for the stack. **By** assuming a characteristic shape for the first mode, it is possible to convert the maximum condition of response into a set of equivalent static forces. The actual analysis may then be

executed on the basis of static analysis (Bl).

The basic concept of the **SEAOC** recommendation is contained in the two formulas

$$
V = KCW
$$
 (4.1)  

$$
C = \frac{0.05}{T l / 3}
$$
 (4.2)

where  $V =$  dynamic shear at base, W *=* total weight of building, <sup>T</sup>*<sup>=</sup>*natural period of first mode, K **=** coefficient reflecting the ability of the structure to deform into the plastic range *(=* **1.5** for brittle structures).

**A** computer program was written to calculate T and W for the stack. The program performs the above calculation to determine the dynamic shear and adds the effect of a **100** MPH wind (22 lb/sq.ft. of Frontal Area) in the same direction as the dynamic shear. The program then calculates the shear stress at **25** different heights of the stack and, because of the stack's circular cross section, the shear stress recorded is increased **by 50%. A** listing of the program is included in Appendix **G.**

The period of the fundamental mode was **1.7** seconds. The calculated results are shown in Table 4.1. Allowable shear stresses for brick stack are given **by** the formula **(M31**

$$
f_{\text{psi}} = 12.3 + 0.037h \qquad (4.3)
$$
  
h = height from top  
(assumes allowable shear stress = 2/3  
allowable working tension)

The allowable stresses are included with the calculated stresses in Table  $4.1$  and in all cases the calculated stresses

#### Table 4.1

### SHEAR **STRESSES** IN **STACK** FROM **100** MPH WIND

#### **AND SEACO** DESIGN **CODE DYNAMIC** SHEAR

## Total DYNAMIC SHEAR at Base = **36,586 # (+SEACO)**



16.5
16.7
16.9
17.2
17.4
17.6
17.8

Table 4.1 Continued

\* Assumes Allowable Shear Stress **= 2/3** Allowable WorkingTension of Brickwork


II REACTOR **BU ILDING 15** MIT BY STACK

are less than the allowable stresses.

### 4.3 WORST **CASE** OF **STACK** FAILURE

In the event that the stack were to collapse, a calculation has been made in which the assumed worst case of stack failure was modeled, and the resultant stresses of the containment building shell roof from the fallen stack were calculated **by** using the SABOR-5 program. 4.3.1 Load Model

The loading model is shown in Figure  $4.2$ . The stack is assumed to be hinged at the base and allowed to fall toward the containment building. Once leaning over the containment building, the stack falls in sections onto the containment building. Section **N** of the stack results in a load in zone **N** on the containment. Zones on the containment building are determined **by** the "shadow" of the stack on the containment building. The mass of stack sections  $\Box$  and  $(2)$  are doubled to take into account the effect of impact. The mass of the stack below **39** feet is not included in the analysis because it cannot hit the containment roof and it could only hit the rigid shield wall.

The loads used in each zone are shown in Table  $4.2.$ The weight of the roof is also used in the stress calculation. The maximum local loading corresponds to about  $5.4$  psi. 4.3.2 Calculational Model

The containment building roof was divided into **23**

# Table 4.2

# **LOADS USED** IN **FALLEN STACK** PROBLEM



Weight of Roof

 $\sim$   $\sim$ 

 $\mathcal{L}_{\mathrm{eff}}$ 

Element **13** use *.158* psiElement 13 use .114 psi

# $FIGURE 4.2$

# MODEL OF FALLEN STACK



 $\infty$ 



 $\mathcal{L}$ 

discrete elements as shown in Figure  $4.3.$  Node  $24$ (corresponding to the top of the concrete shield wall) was considered to be a fixed point. The discrete zone loads were applied to the containment building **by** using the fourier harm6nics **0, 1A,** 1B, **2A,** 2B, 3A, 3B, *4A,* 4B, **5A,** 6A, 7A, and **8A.** The harmonic loading gave loads within **5% of** the actual discrete unaxisymmetric loads.

# *4.3.3* Local Buckling of Roof

The critical pressure (Pcr) of local buckling of a spherical shell is given **by** the equation: (B3) Pcr =  $0.365 \text{ E} (t/R)^2$  (4.4) For our case Roof thickness =  $t = 13/32$  (inches) Shell radius of curvature  $= R = 840$  (inches) Modulus of elasticity =  $E = .3 \times 10^8$  (psi)

This yields a critical pressure of local buckling:

Pcr =  $2.5 \text{ psi}$  (4.5)

Since the fallen stack loading results in equivalent pressure loads of around **5** psi, it appears that the roof will undergo local buckling from the fallen stack.

# 4.3.4 Results of SABOR-5 analysis

The peak stresses on the containment building roof from the fallen stack are given in Table  $4.3$ . These stresses occur on the inside surface of the containment roof. The peak stresses correspond to about one third of the yield point (33,000 psi) and about one fifth of the ultimate strength **(60,000** to **72,000** psi) of the **A-283-C** steel

#### Table 4.3

#### **STRESSES ON CONTAINMENT** FROM **FALLEN** STACK LOADING OF SABOR-5



 $\Im$ 

plate (M2).

Thus **SABOR-5** stress levels indicate that the containment roof will not fracture under the fallen stack loading (this may not hold true if the temperature of the steel is below the ductile transition temperature of  $0^{\circ}$  **F** (M2), which is **highly** unlikely since the building is always heated). Thus it seems that although the shell may buckle under the falling stack, that it is unlikely that the roof will be fractured. The 20 ton polar crane which is supported on thick concrete shield walls should provide a more than adequate means of limiting the buckling of the roof. It appears therefore **highly** improbable that any significant parts of the stack would be able to penetrate through the reactor shielding and cause any damage to the reactor core tank. Although the containment system might no longer be leakproof, the effect of the earthquake will not simultaneously cause a problem in the reactor core for which the containment would be required.

### 4.4 SUMMARY

While there is some probability of the stack hitting the containment  $(-25%)$  if it fails, it appears that even though the containment building may buckle locally, it will not be penetrated **by** a significant portion of the fallen stack. In addition, using the **SEAOC** design and the allowable shear stress for brick stacks, it appears that the

dynamic stress will not be sufficient to cause failure of the stack.

According to Mr. **J.** Fruchtbaum (Office of **J.** Fruchtbaum, Buffalo, New York), who set the design specifications for the stack, special precautions were taken to make the stack very stable; and Mr. Lohr, who was in charge of the construction of the stack, stated that perforated brick was used with liberal amounts of mortar in the joints.

#### Chapter **5**

#### **SUMMARY**

The peak potential horizontal acceleration expected at the MITR-II site is approximately 0.2 **g.** The MITR-II core can be adequately cooled and shutdown so that no major radioactivity release will occur, provided that the core tank remains intact. Analysis of the reactor core tank indicates that much higher accelerations than 0.2 **g** are necessary to cause failure of the core tank. There does not appear to be any significant resonance effect between earthquake motions and the core tank structure or the main coolant pipes, but in any case, it would take a horizontal acceleration of **5.1 g** combined with a vertical acceleration **of** 3.4 **g** before the peak stress of the core tank would equal the yield stress of aluminum. These stresses are far higher than any predictable effect of an earthquake.

While it is conceivable that the effect of an earthquake might cause some damage to the reactor stack, it has been calculated as shown in Section 4.2, that it is **highly** unlikely that peak seismic shear stresses would be above the allowable shear stress of brickwork. It is recommended that the reactor stack be inspected on a regular basis to insure that there has been no deterioration of mortar or brickwork. This inspection process will add assurance that the stack will be able to withstand earthquake motions.

During construction **of** MITR-II, checks should be made that there is adequate lateral restraint of all piping systems. Future penetrations in the reactor building should be made flexible enough to allow for differential earthquake motions.

Temporary shield block walls should not be located where their failure might cause major equipment damage or radioactive release.

Within the scope of this report, it appears that the design MITR-II is adequate to provide required protection even in the event of the maximum expected earthquake motions.

### Appendix **A**

# **SABOR-5 RESULTS** OF THE REACTOR **VESSEL**

- Table A.1 Static Case Comparison of Calculations
- Table **A.2** Comparison of Static Sabor-5 Case with Free-body Diagram
- Figure **A.l** Free-body Diagram of Core Tank
- Table **A.3** Inner Vessel Stress
- Table A.4 Outer Vessel Stress

# Table **A.l**

# STATIC **CASE** COMPARISON OF **CALCULATIONS** (operating condition without  $D_2$ <sup>0</sup> reflector)



# Table **A.2**

# COMPARISON OF STATIC SABOR-5 **CASE** WITH

FREE BODY DIAGRAM

# (Hand Calculation Ref. **15.A-3)**





FIGURE A.I

 $\mathsf{8}^{\mathsf{6}}$ 

Table **A.3**

INNER **VESSEL STRESS**

(Flow Shroud)

Loading Condition

 $\mathbb{R}^d$ 



Static  $+$   $\frac{1}{2}$   $\theta$  sideways  $+$  1/3 up



Station **79**

Station 40

**4e =** shear stress

Station **37**

#### Table **A.4**

#### OUTER **VESSEL** CRITICAL **AREAS**

 $\overline{\phantom{a}}$ 

 $\overline{\phantom{a}}$ 



 $\sim$ 

**mx 00**

# Appendix B

 $\sim$ 

Fortran IV Computer Loads Program used to generate loading input to SABOR-5 for the inner flow shroud.

 $\vec{t}$ 

```
家家聚散社会管理会社科学家聚聚家聚聚聚聚会资源需要要参与营业供应交流合业及票数应为实生资源供应交试点的供应交流的发生应交负责会交易
C
\mathbf cPROGRAM TO CALCULATE INNER VESSEL LOADS
\mathbf CC.
       THERE ARE SIX LOADS ASSOCIATED WITH EACH ELEMENT AND THEY ARE
C.
C
          DENJTED BY
c
              Fl.F2......F6 AND THE REPRESENT
Ċ.
C.
              FI=AXIAL LOAD AT S=0
\mathbf{c}F2=CIRCUMFERENTIAL LOAD AT S=0
Ċ
              F3=NORMAL LOAD AT S=0
C.
              F4=AXIAL LOAD AT S=L
\mathbf{C}F5=CIRCUMFERENTIAL LOAD AT S=L
C.
             F6=NORMAL LOAD AT S=L
\mathbf{c}\mathsf{C}LOADS FOR 4 HARMONICS FOR EACH OF 3 LOADING CONDITIONS ARE
r.
          GENERATED. I.E. FOR GSIDE=0.5.2.5.4.5
\mathbf c\hat{\mathbb{Z}}难要要被最快的激发的要求重新的的动物的大型,如果这样的人的女孩的女孩子的女孩子的女孩子的女孩子的女孩子的女孩子的女孩子的女孩子
       DIMENSION X(100)+PHX(100+2)+T(100+2)+R(100+2)+Y(130)+Z(100+2)+
      1PHI(1)9,2), TIT(20), NX(4), A(9)
\mathbf{C}DENSITY OF WATER
       RH3W=1.93613
\mathbf{c}DENSITY OF ALUMINUM
       RHOA = 0.1r.
      NUMBER OF NODES
       NODES = 80C.
       NUMBER OF ELEMENTS
      NFI = 79N1 = 1\mathbf cREAD AND DEFINE THE GEOMETRY
      READ (4.1000) (X(I).Y(I).I=1,NODES)
      001001=1.797(1.1) = X(1)2(1.2) = X(1+1)R(I,1)=Y(1)120 R(I, 2)=Y(I+1)1900 FORMAT (2E12.6)
      READ (4.1301) (PHX(I.1).PHX(I.2).I=1.NEL)
      00 101 1=1.NEL
      PHI(I.1)=PHX(I.1)*0.0174533
  101 PHI(I.2)=PHX(I.2)*0.0174533
      .00 102 I=1.3
      T(I - 1) = 1.5102 T(1,2)=1.5T(4.1)=1.5T(4, 2) = 2, 5625T(5.1) = 2.5T(5,2)=1.625T(6,1)=1.625T(6, 2) = 1.625T(7.1)=1.625T(7,2)=0.800 103 I = 8, 79T(1.1)=0.75173 T(1,2) = 0.75READ (4.1992) E1.E2.E3.E4.E5
 1002 FORMAT (3X,5E12.6)
 1001 FORMAT (8X.2E12.6)
      NZFRO=0NSABOR=10
      NHARM=4
Ċ.
      START PUTTING JUT THE SABOR5 DATA CONTROL CARDS
```

```
NONE=1
      DO 950 IKL=1.3
      WRITE (7,1003) NZERO, NEL, NODES, NZERO, NHARM, NZERO, NZERO, N1, NSABOR,
     1NZERO.N1
 1003 FORMAT (1114)
      WRITE (6.1004)
 1004 FORMAT (* LANCE TITLE?*)
      READ (5,1005) (TIT(I),I=1,20)
 1995 FORMAT (20A4)
      WRTTE (7.1905) (TIT(I).I=1.20)
      WRITE (7.1003) NZERO
      ZERD = D_0 DWRITE (7.1007) (X(I).Y(I).I=1.NODES)
      07 134 I=1.NEL
      1 = 1 + 1WRITE (7,1006) T.J.PHX(I.1).PHX(I.2)
 1006 FORMAT (214,2E12.5)
  104 CONTINUE
 1007 FORMAT (2E12.5)
     N3=3N7 = 72\mathsf{C}PUNCH THICKNESSES
      WRITE (7,1908) N3, T(1,1), T(1,2)
     WRITE (7,1008) N1, T(4,1), T(4,2)
     WRITE (7.1908) N1.T(5.1).T(5.2)
     WRITE (7,1008) N1, T(6, 1), T(6, 2)
     WRITE (7,1008) N1, T(7, 1), T(7, 2)
     WRITE (7.1008) N7.T(8.1).T(8.2)
 1008 FORMAT (13.2E12.5)
     N8 = 79WRITE (7,1019) N8, E1, E2, E3, E4, E5
 1019 FORMAT (13,5E12.5)
Ċ.
     \mathbf cC.
     END OF GEOMETRY AND CONTROL DATA. BEGIN THE LOAD COMPUTATION
\mathbf{c}LJADS WILL BE GENERATED FOR EACH OF 4 HARMONICS: 0,1,2,AND 4.
C
     ONLY HARMONIC 3 INVOLVES THE UPWARD ACCELERATION
C
\tilde{z}THE FOLLOWING VARIABLES WILL BE USED HEREAFTER
\mathbf cĊ.
        PHI(I.J) = MERIDIONAL SLOPE OF ELEMENT I AT END J
        T(I,J) = THICKNESS OF ELEMENT I AT END J
C.
        Z(I,J) = HEIGHT OF ELEMENT I AT END J
\mathbf cR(I.J) = RADIUS OF ELEMENT I AT END J
\mathbf c\mathbf c\mathbf cHARMONIC<sub>3</sub>
\mathbf{c}C.
     WRITE (7.1003) NZERO.NZERO
     WRITE (6.1009)
\mathbf cC
\mathbf cGET HORIZONTAL ACCELERATION AND HYDROSTATIC HEAD
\mathbf cGSIDE = HORIZONTAL ACCELERATION
c
\mathbf{c}WH = HYDROSTATIC HEAD
        GUP = VERTICAL ACCELERATION = 2/3*GSIDE
r.
\mathbf{c}*********************
                         1009 FORMAT (* GSIDE, WH*)
     READ (5.1000) GSIDE, WH
     GUP=0.666667*GSIDE
     Ē.
C
     ELEMENTS 1 - 3 CONTAIN THE FOLLOWING CONTRIBUTIONS
C
```

```
C.
C.
       (1.)+GUP )*RHOW*(WH-Z) = HYDROSTATIC AND INERTIA LOADS DUE TO
\mathsf{C}WATER
       (1.04GUP)*RHOA*T = GRAVITY AND INERTIA LOAD DUE TO WALL
c.
C.
     \mathbf{c}C.
     F1 = 0.3F2 = 2.0\mathbf{C}F3=(1.0+GUP)*RHJW*(WH-Z(1.1))+(1.0+GUP)*RHOA*T(1.1)
     c.
\mathbf{C}\mathbf{r}F4.F5, AND F6 ARE THE SAVE AS F1.F28 AND F3
C.
C.
     C.
     F4 = 0.9F5 = 0.0\mathbf{c}F6 = F3WRITE (7.1010) N3.ZERO.ZERO.F3.ZERO.ZERO.F6
 1010 FORMAT (13,6E12.5)
     C.
\mathbf{c}ELEMENTS 4 - 7 CONTAIN CONTRIBUTIONS FROM BOTH HORIZONTAL AND
       VERTICAL MOTION
C
\mathbf{c}\mathbf{c}00 105 I=4.7\mathbf{C}最长最容易后要跟我的女家在老师的爱女的程度就会教养善我女家学友女友的女女的女子女子女女的女女女女女女女女子女女子女女子女女女女女女女女女
r.
C.
     F1 = AXIAL LOAD AT S = 2\mathbb C(1.0+GUP)*RHOA*COS(PHI) = GPAVITY AND INERTIA LOAD DUE TO
\mathbf CALUMTNUM
\mathbf{c}c
     F1 = -(1.0+GUP)*RHOA*TI1,1)*COS(PHI(1,1))\mathbf{C}F2 = 0.0\mathbf CF3 = NORMAL LOAD AT S = 6
     C.
C
       D.31831*GSIDE*2.0*RHOW*R = OTH HARMONIC WATER INEPTIA LOAD DUE
C.
\mathbf{C}TO HORIZONTAL MOTION
c
       (1.0+SUP)*RHOW*(WH-Z) = HYDROSTATIC HEAD AND INERTIA LOAD DUE
         TO VERTICAL MOTION
\mathsf{C}(1.7+GUP)*RHOA*T*SIN(PHI) = GRAVITY AND INERTIA LOAD ASSOCIATED
C.
C.
         WITH WALL MATERIAL
C.
     c
     F3=0.31831*GSIDE*2.0*RH3W*R(I.1)+(1.0+GUP)*RHOW*(WH-Z(I.1))+
    1(1.0+GUP)*RHOA*T(I.1)*SIN(PHI(I.1))
            c
\mathbf c\mathbf{C}LOADS F4, F5, AND F6 ARE THE SAME AS F1, F2, AND F3 BUT EVALUATED
\tilde{z}AT S=L
c.
    F4=-{1.0+GUP}*RHOA*T(I.2)*COS(PHI(I.2))
£.
     F5 = 0.0F6=1.31831*GSIDE*2.0*RHJW*R(I.2)+(1.0+GUP)*RHJW*(WH-Z(I.2))+
    1(1.0+SUP)*RHOA*T(1,2)*SIN(PHI(I,2))
    WRITE (7,1010) NONE, F1, ZERD.F3, F4, ZERO.F6
 195 CONTINUE
     N2=2LOADS FOR VERTICAL PORTION OF INNER VESSEL
C
     008811=1.72\ddot{\cdot}\mathbf cTHICKNESS IS CONSTANT IN THIS PORTION
C
    F1 = AXIAL LOAD AT S = 0C.
```

```
C
C.
        (1.)+GJP)*RHOA*T = GRAVITY AND INERTIA LOAD DUE TO WALL
\mathbf{C}MATERIAL
\mathbf cC.
     F1 = (1.0 + GUP) * RH0A * T(9.1)C
     F2 = 2.7C.
     事实社会在社会学校学校教授学校学校科学教授教授学校科学教授学校考试学校研究科技科学科学校教授学校教授学习学校教授学校学校学校学校学校
C.
\mathbf{r}F3 = NORMAL LOAD AT S = 0C
c
       GSIDE#RHOW#2.0*9.31831*R = 9TH HARMONIC COMPONENT OF WATER
C.
          LOAD DUE TO HORIZONTAL MOTION
C.
       (WH-Z)*RHOW = HYDROSTATIC LOAD
C.
\mathsf{C}F3=GSIDE*RHOW*2.0*9.31831*Rf1.11+(WH-Z(I.11)*RHOW
Ċ.
     \mathbf{r}F4.F5. AND F6 ARE THE SAME FORM AS F1.F2. AND F3 EXCEPT THAT
C,
       THEY ARE EVALUATED AT S=L
\mathbf{C}C
C,
     F4=-11.0+GUP1*RH0A*T19.21\mathbf cF5 = 0.0F6=GSIDE*R40W*2.0*9.31831*R(T.2)+(WH-Z(I.2))*RHOW
     WRITE (7.1919) N1.F1.ZER0.F3.F4.ZER0.F6
 881 CONTINUE
     HARMONIC 14
\mathbf{r}\mathbf{c}C
     LOADS FOR HARMONIC 1 ARE DUE TO WALL INERTIA AND THE IST
\mathsf{r}C.
       HARMONIC TERM FROM THE FOURTER SERIES FOR THE WATER INFRTIA
r.
       LOADS
C
\mathbf cWRITE (7.1903) N1.N1
     C
     章家长家家家家的学家家家的家家家的女房家家家的家家的女女女女的女孩的女子家弟会长女女女女女女女女女女女女女女女女女女女女女女女女女女女女女女
\mathbf{c}f.
\mathbf cELEMENTS 1 - 3 INVOLVE ONLY THE WALL LOADS
\mathbf{C}F1 = AXIAL LOAD AT S = nc
\mathbf{C}GSIDE*RHOA*T = WALL INERTIA LOAD
C.
C.
    FI=GSIDE*RHOA*T(1.1)
C.
     F2 = 0.0F3 = 0.0\mathbf c\mathbf{C}F4 IS THE SAME AS F1
     F4=GSIDE*RHOA*T(1.2)
     F5 = 0.0\mathbf cF6 = 0.3C
     WRITE (7,1910) N3,F1,7ER0,ZER0,F4,ZER0,ZER0
C.
     *******
               C
    ELEMENTS 4 - 72 INCLUDE BOTH WATER AND WALL TERMS
\mathbf cC.
\mathbf{C}N4 = 4D0 106 I=4.7
             (李安敬李永承李永敬李永承宋朱永宗李家章李家宗李永子李永亲亲亲亲亲亲亲亲宗宗宗宗宗宗朱永宗宗宗宗宗宗宗宗宗宗宗宗宗
C.
     **********
C.
    F1 = AXIAL LQAD AT S = 0c
\mathbf{c}GSIDE*RHOA*T*SIN(PHI) = WALL INERTIA LOAD
C
```
 $\mathbf{C}$ c FI=GSIDE\*RHOA\*T(I,1)\*SIN(PHI(I,1)) r.  $F2 = 2.3$ c C. C F3 = NJRMAL LOAD AT  $S = 0$ C. C GSIDE\*RHOA\*T\*COS(PHI) = WALL INERTIA LOAD C. GSIDE\*2.0\*RH3W\*R\*3.5 = IST HARMONIC CONTRIBUTION OF WATER ſ. **INERTIA** C. c F3=GSIDE\*RHDA\*T(I.1)\*COS(PHI(I.1))+GSIDE\*2.3\*RHOW\*R(I.1)\*9.5 **木束学和自己的家长和学校的女士和教育的家长在教育家会长大家的家长开餐厅多个家具的书头中一个大的天地的女士和女主教会会长手套女主教会会** C c F4.F5. AND F5 ARE THE SAME FORM AS F1.F2. AND F3 BUT EVALUATED AT C.  $S=1$ C, **章家只要恢复李家康荣豪荣誉荣誉荣誉荣誉荣誉荣誉荣誉荣誉荣誉汉海棠的发布**为孩子的未来要跟为老老童童文学大学大学教授教育文学学校文学大会 c C.  $F5 = 0.0$ F6=GSIDE\*RHOA\*T(I+2)\*COS(PHI(I+2))+GSIDT\*2+0\*RHOW\*R(I+2)\*0+5 WRITE (7.1010) NL.F1.ZERO.F3.F4.ZERO.F6 196 CONTINUE  $\mathbf c$ c  $\mathbf{c}$ FOR ELEMENTS 8 - 72 THE LOADS ARE SAME AS 4 - 7 BUT WITH PHI=3.0 r. AND T=CONST. ALSO NOTE THAT F3=F6 C. c C.  $F1 = 0.7$ ۰C  $F2 = 0.0$ F3=GSIDE#RHDA\*T(8+1)+GSIDE\*®+5\*2+C\*R(8+1)\*RHOW ſ.  $F4 = 0.0$ ſ.  $F5 = 0.0$  $F6 = F3$ WRITE (7,1919) N7, ZERO, ZERO, F3, ZERO, ZERO, F6  $\mathbb C$ C c FOR HARMONICS 2 AND 4 THE ONLY CONTRIBUTIONS ARE FROM THE WATER  $\mathbf c$ **INERTIA TERMS**  $\mathbf c$ C. c. HARMONIC 2A WRITE (7.1003) N2.N1 WRITE (7.1910) N3.ZERO.ZERO.ZERO.ZERO.ZERO.ZERO.ZERO  $00117$   $1=4.7$ F3=0.21221\*GSIDE\*2.0\*R(I.1)\*RHOW F6=3.21221\*GSIDE\*2.0\*R(I,2)\*RHOW WRITE (7.1910) N1.7ERO.ZERO.F3.ZERO.ZERO.F6 **107 CONTINUE** F3=0.21221\*GSIDE\*2.0\*R(8.1)\*RHOW WRITE (7.1910) N7. ZERO. ZERO.F3. ZERO, ZERO.F3 HARMONIC 4A  $\mathbf{c}$ WRITE (7,1303) N4, N1 WRITE (7.1010) N3. ZERO. ZERO. ZERO. ZERO. ZERO. ZERO  $001081 = 4.7$ F3=-J.J4244\*GSIDE\*2.0\*R(I.1)\*RHOW F5=-3.34244\*GSIDE\*2.0\*R(I.2)\*RHOW WRITE (7,1010) N1,ZERO,ZERO,F3,ZERO,ZERO,F6 **108 CONTINUE** F3=-. 04244#GSIDE\*2.0\*R(8.1)\*RHOW WRITE (7.1010) N7.ZERO.ZERO.F3.ZERO.ZERO.F3 SABOR 5 PORTION C. WRITE (6.1011) 1011 FORMAT (\* SABOR5 TITLE?\*)

```
READ (5,1905) (TIT(I), I=1,29)
   WRITE (7.1005) (TIT(I).I=1.20)
   WRITE (7.1903) N1.NZERO.NZERO.N1.NZERO.NZERO.NZERO.N1
   WRITE (7,1903) NZERO, NZERO, N2, NZERO
   NO = 0N9 = R9N33 = 3N4=4WRITE (7,1003) NO.NO.NO.NO.NO.NO.NO.NO.NO.NO.NO
   WRITE (7.1003) N2.N4
   WRITE (7,1903) N4, N9, N33
   WRITE (7.1903) N1.N1.N2.N2
   WRITE (7.1003) NO.NO.NO.NO.NO.NO.NO.NO.NO
   WRITE (7.1003) N2.N4
   WRITE (7.1003) N4.N9.N33
   RING LOAD DUE TO TREATMENT OF FUEL FLEMENTS AND ASSOCIATED
      STRUCTURE AS LUMPED MASS. COMPUTATION VIELDS FACT THAT THIS
      CONTRIBUTION IS INSIGNIFICANT EVEN FOR LARGE GSIDE. SEE NOTES
   F1=300.0/(2.0*3.141593*R(9.1))*GSIDE
   F3=300.0/(2.0*3.1415931*GSIDE
   WRITE (7.1906) N1.N1.F1
   WRITE (7,1006) N1, N4, F3
   D0 199 I=2.4.2
   WRITE (7,1903) I.N1.N2.N0
   WRITE (7.1003) NO.NO.NO.NO.NO.NO.NO.NO.NO.NO.NO
   WRITE (7.1003) N2.N4
   WRITE (7,1003) N4, N9, N33
199 CONTINUE
950 CONTINUE
   CALL EXIT
   FND
          TWO CARDINAL WAY CITY
                                  \sim 10^{-1}
```
 $\sim$ 

 $\mathbf{c}$  $\mathbf c$ 

 $\mathfrak{c}$ 

 $\mathbf{C}$ 

 $\mathbf c$  $\mathfrak{c}$ 

 $\mathsf{C}$ 

 $\sim$   $\sim$ 

 $\lambda$ 

 $\bar{z}$ 

 $\mathbf{r}_\mathrm{c}$ 

 $\sim 10^{-1}$ 

Fortran IV Computer Loads Program used to generate loading input to SABOR-5 for the outer core tank.

 $\sim$ 

```
C
C PROGRAM TO CALCULATE OUTEP VESSEL LOADS
C
C THERE AlE SIX LOADS ASSOCIATED WITH CACH ELEMENT AND THEY ARE
C DENOTED BY<br>C Fl.F2...
            C Fl.F2,...,F6 AND THE REPRESENT
C
C F1=AXIAL LOAD AT S=0<br>C F2=CIRCUMFERENTIAL L
C F2=CIRCUMFERENTIAL LOAD AT S=0
C F3=NORMAL LOAD AT S=0
C<br>CF4=NORMAL LOAD AT S=L<br>
CF4=NORMAL LOAD AT S=L
            C F6=NORMAL LOAD AT S=L
C
C LOADS FO TWO HARMONICS FOR EACH OF THE CONDITIONS AE
      C GENERATED
C
      DIMENSION X(100),PHI(100,2),T(100,2),R(1v2,2,Y(10),Z(100,2),
     ITIT(120),NX(4),IHAR(4),FACT(4),A(10)
      DIMENSION JHAR(4)
      READ (5.2884) NTIME. (JHAR(I), I=1.4), NTHET, NSTOP
 2884 FORMAT (7141
C DFNSITY OF WATER IN LBS/(CUBIC IN)
      RHOW=0.03613
C DENSITY OF ALUMINUM IN LBS/(CUBIC IN)
      RHOA=0.1
C MASS DENSITY OF ALUMINUM
      XMASS=RHOA/32.2
C NUM8ER OF NODES
      NODE S=95
C NUMBER OF ELEMENTS
      NEL=94
      NI=3
C MISCELLANEOUS CONSTANTS USED FOR CONTROL OF SABOR5
      NO=JHAI(1)+JHAR(2)+JHAR(3)+JHAR(4)
      FACT(11=0.5/13141593
      FACT(2)=1.0/3.141593FACT( 3)=FACT(2)
      FACT(41=FACT(3)
      N1=1
      IST=3
     N4=4
      N10=10
     N94=94
     ZERO=0.0N2=2
     N4=4
     N83=83
     N12=12
     PHIREF=1.57080
 1010 FORMAT (2E12.5)
C *********i**************'**************** *************
C
     C FROM HERE TO POINT NOTED READ IN GEOMETRY AND MATERIAL PROPERTIES
C
C
     READ (5,1000) (X(I).Y(I).I=1,NODES)
     DO 110 I=1.NEL
     Z(I,1)=X(T)
     Z(I, 2) = X(I+1)R(I.1)=Y( I)
```

```
100 R(I, 2)=Y(I+1)1000 FORMAT (2E12.5)
 1005 FORMAT (20A4)
      READ (5,1001) (PHI(I,1), PHI(I,2), I=1, NEL)
      0010111=1.8READ (5.1002) N.T1.T2
 1002 FORMAT (13.2E12.6)<br>1001 FORMAT (8X.2E12.5)
      IBFG=IST+1TEND=IST+N
      03 102 J=IBEG.IEND
      T(J,1)=T1102 T(j,2)=T2
  1^1 IST=IST+N
      READ (5.1043) E1.E2.E3.E4.E5
 1043 FORMAT (3X.5E12.5)
\overline{\mathbb{C}}未要整整数数数本本部整整法决策数据数据数据数据数据数据数据数据数据数据数据数据数据数据数据数据数据表示对方 计数据数据 化二乙基丙烯
\mathbf{r}C.
      END OF GEOMETRY INPUT
\mathbf{c}LOOP 950 CONTROLS CALCULATION FOR THE NUMBER OF CASES DESIRED
\mathsf{C}c.
         NORMALLY NTIME = 3 FOR THE THREE LOADS GSIDE=0.5.2.5.4.5
\mathbf{C}\mathbf{c}WRITE (6,1009)
 1009 FORMAT (* GSIDE, WH?')
      C
C.
Ċ,
      GET HORIZONTAL ACCELERATION AND HYDROSTATIC HEAD
C.
r.
         GSIDE = HORIZONTAL ACCELERATION
\mathbf cWH = HYDROSTATIC HEAD
c
         GUP = VERTICAL ACCELERATION = 2/3*GSDE\mathbf{C}\mathbf cREAD (5,1010) GSIDE.WH
ŗ.
      PUNCH GEOMETRY AND SABOR5 CONTROL DATA
      GUP=1.666667*GSIDE
      WRITE (7.1003) NO.NEL.NODES.NO.NO.NO.NO.NI.N10.NO.NO
 1003 FORMAT (1114)
      WRITE (6,1004)
 1004 FORMAT (* LANCES TITLE?*)
     READ (5.1005) (TIT(I).I=1.20)
     WRITE (7.1005) (TIT(I).I=1.29)<br>WRITE (7.1003) NO
      WRITE (7.1000) (X(I).Y(I).I=1.NODES)
      I = 1J=2WRITE (7.1006) I.J.PHI(I.1).PHI(I.2).N94
 1006 FORMAT (214.2E12.5.8X.14)
     00 103 1=2.83J = I + 1WRITE (7.1006) I.J.PHI(I.1).PHI(I.2)
 103 CONTINUE
     K = 1J = 851 = 84\epsilonWRITE(7.1006) K.J.PHI(I.1).PHI(I.2)
     00 4147 I = 85.94J=I+1WRITE(7,1006) I.J.PHI(I.1), PHI(I.2)
4147 CONTINUE
     WRITE (7,1007) (N1, T(I,1), T(I,2), XMASS, I=1, NEL)
1007 FORMAT (13,3E12.5)
     WRITE (7,1008) NEL, E1, E2, E3, E4, E5
1008 FORMAT (13.6E12.5)
```

```
\mathbf{c}TERMINATE IF ONLY GEOMETRY WANTED
     IF (NSTOP .NF. 0) CALL EXIT
\mathbf{c}C.
\mathbf{c}END OF GEOMETRY AND CONTROL DATA. BEGIN THE LOAD COMPUTATION
     LOADS WILL BE GENERATED FOR EACH OF 4 HARMONICS: 0.1.2.AND 4.
c
c
     ONLY HARMONIC O INVOLVES THE UPWARD ACCELERATION
\mathbf cC,
     THE FOLLOWING VARIABLES WILL BE USED HEREAFTER
\mathbf{c}c
        PHI(I.J) = MERIDIONAL SLOPE OF ELEMENT I AT END J
C
        T(I.J) = THICKNESS OF ELEMENT I AT END J
c
        Z(I.J) = HEIGHT OF ELEMENT I AT END J
        R(I.J) = RADIUS OF ELEMENT I AT END J
\mathbf c\mathbf cC.
     HARMONIC O
ſ.
c.
     WRITE (7,1906) NO.NO
     TF (JHAR(1) .EQ. 0) GO TO 176
     DO FOR ALL ELEMENTS
\mathbf{c}NFT = 1NEL = 34NIR = -18592
     CONTINUE
     DO 104 I=NET.NEL
\mathbf c******
                 C
\mathbf{c}F1 = AXIAL LOAD AT S = 0C
        (1.)+GUP)*RHOA*T*COS(PHI) = GRAVITY AND INERTIA LOAD ASSOCIATED
\mathbf cc
          WITH WALL MATERIAL
\mathbf cC.
     F1=-R+0A*(1.0+GUP)*T(I,1)*COS(PHI(I,1))
C
     F2 = 0.0\mathbf{c}C
r.
     F3 = NORMAL LOAD AT > = 0 AND IS COMPOSED OF THE FOLLOWING TERMS
C
C
       (WH-Z)*RHOW = HYDROSTATIC WATER LOAD
c
       (WH-Z)*GUP*RHOW*SIN(PHI) = WATER INERTIA LOAD DUE TO
C
          VERTICAL MOTION OF VESSEL
       (1.0+GUP)*SIN(PHI)*RHOA = INERTIA AND GRAVITY LOAD DUE
\mathbf{r}C.
          TO VERTICAL MOTION OF THE WALL
\mathbf cc
     F3=(dH-Z(I.1))*RHOW+(WH-Z(I.1))*GUP*RHOW*SIN(PHI(I.1))+(1.3+GUP)*
    1 T(T.1) & STN(PHT(T.1) ) *RHOA
\mathbf{C}C.
     THE LOADS F4,F5, AND F6 CONTAIN THE SAME CONTRIBUTIONS AS DO
C.
C.
       F1.F2. AND F3. BUT ARE EVALUATED AR S=L OF THE ELEMENT
\mathbf c\mathbf{c}F4=-R40A*(1.0+GUP)*T(I.2)*COS(PHI(I.2))
\mathbf{c}F5 = 0.0F6=(WH-Z(I,2))*RHOW+(WH-Z(I,2))*GUP*RHOW*SIN(PHI(I,2))+(1,0+GUP)*
    1T(I.2)*SIN(PHI(I.2))*RHOA
     WRITE (7,1008) N1, F1, ZERO.F3, F4, ZERO.F6
 104 CONTINUE
     IF(NIR) 8593.8594.8595
8593 NET=35
     NEL = 37NIR=0
     RHOA=B_06
```

```
GO TO 8592
                                                             \sim8594
     NET=38
     NEL = 94\mathbf{c}NIR=10RHNA=0.1
     G7 TO 8592
8595
     CONTINUE
\mathbf{c}c
     IN THE FEET AREA OF THE FEET. THE FEET ARE TREATED AS A DISTPIBUTED LOADIN
c
     IN THE FEET AREA OF THE FEET, THE FEET ARE TREATED AS
\mathbf{C}DISTRIBUTED LOADING BY INCREASING THE DENSITY OF THE TANK
C,
\mathbf c\mathbf{C}C.
C
     HARMONTC 1A INVOLVES ONLY SIDEWAYS ACCELERATIONS
C.
C.
C
     176 IF (JHAR(2) .EQ. 0) GO TO 8599
     WRITE (7.1006) N1.N1
\mathbf{r}FOR ALL ELEMENTS
     NFT=1NEL = 34NIR=-18596
     CONTINUE
     00 105 I=NET.NEL
\mathbf{C}C
C.
     LOADS F2 AND F5 ARE BOTH ZERO. THE GRAVITY LOAD EXISTS ONLY FOR
c
        HARMONIC 3. LOADS IN THE AXIAL AND NORMAL DIRECTION DEPEND ONLY
\mathbf cON THE LOCAL MERIDIONAL SLOPE AND ARE DUE ONLY TO THE WALL
Ċ.
        MATFRIAI
c
\mathbf{C}F1=GSIDE*RHOA*T(I.1)*SIN(PHI(I.1))
     F3=GSIDE*RHOA*T(I,1)*COS(PHI(I,1))
     F4=GSIDE*RHDA*T(I,2)*SIN(PHI(I,2))
     F6=GSIDE*RHOA*T(I.2)*COS(PHI(I.2))
     WRITE (7,1008) N1, F1, ZERO, F3, F4, ZERO, F6
  105 CONTINUE
     IF(NIR) 8597,8598,8599
8597 NFT=35
     NFI = 37NIR = 0RHOA=19.1
     GN TO 8596
8598
     NET=38
     NEL = 94RHOA=0.1
     NIR = 10GN TO 8596
8599 CONTINUE
c.
     SABOR5
 179 WRITE (6,1011)
1011 FORMAT (' SABOR5 TITLE?')
     READ (5.1005) (TIT(I).I=1.20)
     WRITE (7.1005) (TIT(I).I=1.20)
     WRITE (7,1903) N1, N1, N9, N1, N9, N9, N9, N1
     IHAR(I)=0IHAR(2)=1IHAR(3)=2IHAR(4)=4NX(1)+1NX(2)=1
```
 $\mathcal{L}$ 

 $\bar{z}$ 

```
\bar{\lambda}NX(3)=1NX(4)=1N8=800106 1=1.4IF (JHAR(I) .EQ. 0) GO TO 106
        NQ = QWRITE (7+1003) IHAR(I)+NX(I)+N1+N0<br>WRITE (7+1003) ND+N0+N0+N0+N0+N0+N0+N0+N0
        WRITE (7,1003) N1,N83
106
        CONTINUE
         C.
        FROM HERE TO END IS CONTROL DATA FOR THE PLOT PROGRAM
c
\mathbf c\ddot{\rm c}DO 210 I=1.NTHET
   210 A(I)=45.0*(I-1)
        WRITE (7.1003) NTHET.NO.NO
        WRITE (7,1022) (A(I), I=1, NTHET)
 1022 FORMAT (8F10.3)
        A(1)=0.0A(2) = 0.5A(3)=1.0DO 211 I=1, NTHET
        WRITE (7,1003) NO.NO.N3
        WRITE (7.1023) (A(J).J=1.3)
 1023 FORMAT (3E12.5)
   211 CONTINUE
   950 CONTINUE
                                                   \simCALL EXIT
        END
                                  \mathcal{L}^{\mathcal{L}} . The contract of the \mathcal{L}^{\mathcal{L}}\sim 100 km s ^{-1}\mathcal{A}^{\mathcal{A}} and \mathcal{A}^{\mathcal{A}}\sim 10^{-11}\sim 10^{-10} km
                  Standard Stand
```
 $\sim$ 

 $\bar{z}$ 

 $\sim 10^7$ 

# Appendix **D**

Fortran IV Computer Program used in Determination of Fundamental Frequency of the Core Tank using the Method of Vianello and Stodola **(H3).**

Figure **D.l** Assumed Initial Displacements

```
ſ.
PROGRAM TO CETERMINE TE FUNDAMENTAL FREQUENCY OF THE CORE TANK
\mathbf cDIMENSICN T(4,4), A(4,4), XX(4,4)
       DIMENSION C(55), F(95, 95), AP(95)
       DIMENSICN E(4,4), XY(4,4)
       DIMENSION IWORK(95), JWORK(55), XMAS(95,55)
       DIMENSION WAT(95)
       DIMENSION X(95,95), PHI(100)
\mathbf c\mathbf c\mathsf{C}\mathbf{C}THE MASS MATRIX AND THE STIFFNESS MATRIX ARE OBTAINED FROM
\mathbf cTHE SABOR 5 PROGRAM AND CCRESPEND TO THE GEOMETRY USED IN
\mathbf cSTRESS DETERMINATIONS
\mathbf{c}X(I,J) = STIFFNESS MATRIX\mathsf{C}XMASS(I,J) = MASS MATRIX\mathbf cC
       CALLING ON THE STIFFNESS MATRIX FROM SABOR 5
       CALL READIN
10FCRMAT (5F1C.5)
11
       FORMAT (6E13.6)
       FORMAT(15X,28HX-DIRECTION STIFFNESS MATRIX )
12\mathsf{C}READING IN THE VALUES OF PHI
       00 410 I=1,55DC 410 J=1,95XMAS(I, J) = 0.0410X(1, J) = 0.0REAO(5,10) (PHI(J), J=1,95)\mathbf c\mathbf cC
       SABOR 5 DJES NCT STORE THE FULLY POPULATED MATRIX BUT ONLY
\mathbf cSTORES THE NEN-ZERO TERMS
\mathbf cTHE FOLLOWING STEPS ARE TO TO TAKE THE DESIRED COEFFICIENTS
\mathbf cFRCM THE SABOR MATRIX
C
C
\mathbf c\ddot{\cdot}\mathbf{C}C
\mathbf cTHE CCORDINATE SYSTEM MUST BE CHANGED FROM THE SABOR SYSTEM TO
\mathbf cAN X - Y - Z SYSTEM
\mathbf cDETERMINING THE STIFFNESS MATRIX
\mathbf cCALL ERASE (T,16)
       00 100 I=1,94
       CSI=CCS(PHI(1))SNI = SIN(PHI(1))J = I + 1CALL ERASE (A,16)
      CALL ERASE (8,16)
      CSJ=CCS(PHI(J))
       SNJ = SIN(PHI(J))T(1,1) = CSI
       T(2,1) = -SNIT(1,2) = SNI
       T(2,2)=CSIT(3,3) = CSJT(3,4) = SNJT(4,3)-5NJT(4,4)=CSJCALL TERM (1,1,1,1,8(1,1),A(1,1))CALL TERM (I,1,I,3,B(1,2),A(1,2))
```

```
CALL TERM (1,3
,I,3,8(2,2),A(2,2))
      CALL TERM (J,1,1,1,8(3,1),A(3,1))
       CALL TERM (J,1
,1,3,8(3,2),A(3,2))
       CALL TERM (J,1
,J,1,B(3,3),0(3,3))
       CALL TERM (J,3,I,1,B(4,1)
       CALL TERM (J,3
,I,3,8(4,2),A(4,2))
       CALL TERM (J,3
,J,1,B(4,3),h(4,3))
       CALL TERM (J,3
,J,3,B(4,4),A(4,4))
      DO 40C K=1,4
      DO 400 L=1,K
      B(L,K)=B(K,L)400 A(LK)=A(KL)
      00 401 11=1,4
      00 401 JJ=1,II
      SUMq=C.0
      SUMA=C .0
      00 402 KK=1,4
      00 402 LL=1,4
      SUMB= SUMB+T(KK,II)*B(KK,LL)*T(LL,JJ)
402 SUMA=SUMA+T(KK,II)*A(KK,LL)*T(LLJJ)
      XY(II,JJ)=SUMA
      XY(JJ,1I)=SUMA
      XX(I1 ,JJ)=SUPB
401 XX(JJ,II)=SUMB
      IF (I *NE. 1) GO TO 640
      X(1,1 )=XX(1,1)
      XMAS( 1,1)=XY( 1,1)
640 X(1,1)=XX(1,3)<br>X(1,1)=XX(3,1)X(J,J)=XX(3,3)XMAS(I, J) = XY(1, 3)XMAS(J,T)=XY(3,1)XMAS(J,,J)=XY(3,3)
100 CCNTTILF
      n 4n' 1=1,S9
      DO 40F .=C2,95
      XMAS( IJ)=f0.0
      XMAS( J, 1)=C,.1
      X(I,J)=1.n
4050 X(J,I )=.0
      DO 4051 J=92,95
      XMAS(J,J)=).0
4051 X(JJ)=1.0
      Do 101 I=1,S5,10
      IST-I
      IED=MIA0(I+9,95)
 3345 FORMAT(1X,10E12.4)
      WRITE (f,3345) ((X(K,J)
,J= ISTIED) ,K=1,95)
 1001 CONTINUE
C
C
C
C
      INVERUTNG THE STIFFNESS MATRIX YIELCS THE FLEXIBILITY MATRIX
      ABG=0.0
      DO 80C 1=1,95
  800 ABG=ABG+ALOG(X(I,I))
      ABG=ABG/95 .0
      ABG=EXP(ABG)
      A8G=1 *0/A BG
      DO 801 1=1,95
      DO.801 J=1,95
  801 X(1,J)=X( I,J)*ABG
      CALL MINV (X,95,D, IWCRK, JWCRK)
      00 802 T=1,95
      DO 802 J=1,c5
  802 X(I,J)=X( I,JI*ABG
```
2245 **FCRMAT (14) FORMAT (5E12.6)** 2246 READ(5,2245) JOT 2247 FORMAT (5X,14,5X,36HFUNCAMENTAL FRECUENCY OF CORE TANK ) WRITE(6,2247) JOT  $\mathbf c$  $\mathbf c$  $\mathbf c$ AN INITIAL SET OF DISLPACEMENTS IS ASSUMED  $\mathbf c$ C  $\mathsf{C}$ READING IN THE ASSUMED DISPLACEMENTS  $\mathbf C$  $\mathbf{C}$  $\mathbf c$ THE MASS MATRIX FROM THE SABOR PROGRAM ONLY INCLUDED THE C MASS OF THE TANK, THUS THIS MASS MUST BE INCREASED FOR THE C MASS OF THE WATER BY INCLUDING LUMPED MASSES AT EACH NODE C С THE FACTOR OF 32.2 OCCURS BECAUSE SABOR 5 GIVES THE MASS MATRIX  $\mathsf{C}$ IN SLUGS  $\mathbf{C}$  $\mathsf{C}$  $READ(E, 2246)$  (C(I),  $I=1,55$ )  $2 = 14.0$ DC 1000 J=19,40 WAT(J)=0.833\*0.833\*0.0833\*3.14\*62.4/32.2  $XMAS (J, J) = XMAS (J, J) + WAT (J) / 2.0$  $1000$  XMAS(J+1,J+1)=XMAS(J+1,J+1)+WAT(J)/2.0 DO 1100 J=41,46 WAT(J)=0.833#0.833#3.14\*0.0333\*62.4/32.2  $XMASL$  J, J) =  $XMASL$  J, J) +  $hAT$  (J) /2.0 XMAS(J+1, J+1)=XMAS(J+1, J+1)+WAT(J)/2.0  $1100$  $00 10C2 J=47,60$ WAT(J)=(0.833+1.0/Z)\*\*2.0\*C.062\*3.14\*62.4/32.2  $2 = 2 - 1.0$  $XMAST$  J, J) =  $XMAST$  J, J) +  $WATT$  (J) /2.0  $1002 XMAS$ (j+1,j+1)= $XMAS$ (j+1,j+1)+ $WAT$ (j)/2.0  $00 1003 J=61,65$ WAT(J)=1. E7\*1. 87\*0.145\*3.14\*62.4/32.2  $XMAS$ ( J, J) = XMAS( J, J) + WAT( J)/2.0  $1003$  $XMAS$ ( J+1, J+1)=XMAS( J+1, J+1)+WAT( J)/2.0 xmAS(E6,66)=xmAS(66,66)+(1.87\*1.87\*0.30\*3.14\*62.4/64.4) XMAS(67,67)=XMAS(67,67)+(1.87\*1.87\*0.30\*3.14\*62.4/64.4)  $00 1004 J=67,71$ WAT(J)=1.87\*1.87\*3.14\*C.163\*62.4/32.2 O.S. (L) TAW+(L,L) 2AMX=(L,L) 2AMX 1004 XMAS(J+1, J+1)=XMAS(J+1, J+1)+WAT(J)/2.0  $00 1005 J = 72,92$ WAT(J)=2.083\*2.083\*0.145\*3.14\*62.4/32.2  $XMAS (J, J) = XMAS (J, J) + WAT (J) / 2.0$  $1005$ XMAS(J+1, J+1)=XMAS(J+1, J+1)+WAT(J)/2.0  $001006$   $J=53,94$ WAT(J)=2.083\*2.083\*3.14\*0.33\*62.4/32.2  $NMS(1,1)$  TAW+(L,L)2AMX=(L,L)2.0 1006 XMAS(J+1,J+1)=XMAS(J+1,J+1)+WAT(J)/2.0 XMAS(48,48)=XMAS(48,48)+750.0/32.2  $XMAS(49,49)=XMAS(49,491+750,0/32,2)$ XMAS(50,50)=XMAS(50,50)+75C.0/32.2  $\mathbf{C}$  $\bar{\alpha}$ C THE MASS MATRIX AND THE FLEXIBILITY MATRIX ARE MULTIPLIED  $\mathbf c$  $\mathsf{C}$ TOGETFER WITH THE ASSUMED CISPLACEMENTS TO GIVE A LAMBDA AND A NEW SET OF DISPLACEMENTS  $\mathbf{C}$  $\mathbf c$ CALL ERASE (AP, 95)

CALL ERASE (F,9025)  $00 101 I = 1,95$ 

```
00101 J = 1.1SUM = 0.0DO 75C K=1,95
750
       SUP=SUM+32.2*X(I,K)*XMAS(K,J)
       F(I,J)=SUM101F(J, I) = SUM2248
       FORMAT(2X, 10HL AMBCA = , E12, 6)
1050FORMATISX,4HNODE,10X,11HLUMPED MASS,10X,15HX-DISPLACEMENT )
1051
       FORMAT(5X, I4, 9X, E12.6, 12X, E12.6)
       L = 100II=0200C = C \cdot 000 102 J=1,5500 2005 I = 1,55AP(J) = AP(J) + (F(J, I) * D(I))2005 CCATINUE
       IF(AP(J)-C) 201,201,202
202
       C = AP(J)CONTINUE
201102CONTINUE
       II = II + 1901
       0030C J=1,55D(J) = AP(J)/C\mathbf cC
       THE PROCESS IS ITERATES 100 TIMES
\mathbf cC = LANDDAR = CMEGAC\mathbb{C}\mathbf c300
      CONTINUE
       WRITE (6,2248) C
       IF(L-II) 900,900,200900
       R = SQRT(1.0/C)950
       FORMAT (2X, 30+CMEGA FOR THE MITR-2 VESSEL IS, 2X, E12.6)
       WRITE(6,950) R
       WRITE(6,1050)
       DC 10C9 J=1,95WRITE(6,1051) J,XMAS(J,J),C(J)
1009 CENTINUE
       CALL EXIT
       FND
       SUEROUTINE REACIN
       DIMENSICN XK(2454), XM(2454), NCOL(400)
\mathbf cSUBROUTINE READIN
                            OBTAINS THE THE STIFFNESS AND MASS
C
\mathbf{C}MAIRICES FROM THE STORAGE LOCATION OF THE SABOR 5
\mathbf{C}PROGRAM USED IN RUNNING THE STRESS ANALYSIS
\mathbf c\mathbf cINTEGER#4 C1, D2
       DC 100 1=5,380,4J = I - 4NCCL(I)=J
       NCCL(I+1) = J
       NCCL(1+2) = J
       NCCL (1+3) = J
      WRITE (6,4000) I, NCOL(I), NCOL(I+1), NCOL(I+2), NCOL(I+3)
 4000 FORMAT (5110)
  100 CONTINUE
      D0 101 1=1,4101 NCOL(I)=1
       READ (8,1000) (XM(I), I=1, 2454)
 1000 FORMAT (6F13.6)
       READ (8,1000) (XK(I), I=1, 2454)
       RETURN
       ENTRY TERM (N1, D1, N2, D2, VALUE1, VALUE2)
       IRC W1 = (N1 - 1)*4 + 01
```
 $\ddot{\phantom{0}}$  $IRCW2 = (N2 - 1)*4 + D2$ IRCW=MAXO(IRCW1,IRCW2) ICCL=MINO (IRCW1, IROW2) INDEX=ICOL IF (ICOL .GE. ACOL(IROW) .AND. ICOL .LE. IROW) GO TO 103 WRITE (6,1001) N1,01,N2,D2<br>1001 FORMAT (/\* THE VALUES N1,D1,N2,D2 = \*,3(I4,\*,\*),I4,\* FORM AN INVAL<br>1ID COMBINATION, VALUE1 AND VALUE2 ARE SET TO 0.0\*)  $VALUE1=0.0$ VALUE  $2 = 0$ . 0 **RETURN** 103 00 102 I=1, IRCW 102 INDEX=INDEX+I-NCOL(I)

 $\ddot{\phantom{a}}$ 

VALUE1=XK (INDEX) VALUE2=XM(INDEX) RETURN ENC.

 $\sim$ 

 $\sim$ 

 $\mathcal{L}$ 

 $\sim$ 

c


## Appendix E

## **CALCULATION** OF **FUNDAMENTAL** MODE OF **COOLANT** PIPE

The main light water coolant pipe is the longest unrestrained major pipe. The pipe will be constrained against large motions because of the tightness of the area through which it passes.

The dimensions of the pipe are:

$$
D_0
$$
 = outside diameter = 8.5 inches  
\n $D_i$  = inside diameter = 8.0 inches  
\n $\ell$  = length = 15 feet  
\n $I$  = Moment of inertia =  $\pi (D_0^4 - D_i^4) = 56 inches$ <sup>4</sup>

 $E$  = modulus of elagticity of aluminum =  $0.1$  **x**  $10^8$ pounds/ inches

the mass of the pipe and water is:

$$
m = \pi((D_0/2)^2 - (D_i/2)^2) (\mathbf{R}) (\mathbf{P}_{A1}) + \pi (D_i/2)^2
$$
  

$$
\mathbf{Q} (\mathbf{P}_{H20})
$$
  
(E.1)  
where  $\mathbf{P}_{A1}$  = density of aluminum = 0.1 pounds/inch<sup>3</sup>  
 $\mathbf{P}_{H_20}$  = density of water = 0.036 pounds/inch<sup>3</sup>

thus

 $m = 442$  pounds

Now assuming the coolant pipe acts as a circular pipe with fixed ends, the fundamental mode is:

$$
w = \frac{25 \pi^2}{16 \text{ R}} \sqrt{\frac{E I}{m}}
$$
 (E.2)

thus

 $\sim 10^{-1}$ 

w = 
$$
\frac{25}{16} \left( \frac{3.14}{12 \times 15} \right)^2
$$
  $\sqrt{\frac{(0.1 \times 10^8)}{(442)}}$  (E.3)

## Appendix F

# **CALCULATION** OF GUIDE **TUBE** DISPLACEMENT

## FROM **1 G** HORIZONTAL ACCELERATION

Referring to Figure **3.5,** the displacement of a control guide tube of length  $\ell$  is to be determined at distance X from where  $\hat{V}$  equals  $0$ . The guide tube is modeled as a cantilever beam which is fixed at  $\hat{\chi} = 0$ .

For a cantilever beam under a distributed loading, the displacement at X is **:** (Ml)

$$
X = \frac{w}{24 ET} (X4 - 4R3 X + 3R4)
$$
 (F.1)

For a cantilever beam with a concentrated load  $at\beta$ , the displacement at X is: (Ml)

$$
X = \frac{P}{6 \text{ ET}} (2 \, \text{R}^3 - 3 \, \text{R}^2 x + x^3) \qquad (F.2)
$$

where

**E =** modulus of elasticity

<sup>I</sup>**<sup>=</sup>**moment of inertia of the guide tube

 $P =$  concentrated load at length =  $Q$ 

<sup>w</sup>**<sup>=</sup>**distributed load (pounds/inch)

The mass of the guide tube acts as a distributed load during a horizontal acceleration, and the mass of the guide rod and blade is assumed to act as a concentrated load at  $\oint$  during a horizontal acceleration. For a **1 G** acceleration, the loads equal the respective gravitational mass of the guide tube and rod.

Now

$$
x = 31 inches
$$
\n
$$
R = 48 inches
$$
\n
$$
E = .1 \times 10^8 pounds/inches^2
$$
\n
$$
D_0 = outside diameter of guide tube = 3 inches
$$
\n
$$
D_i = inside diameter of guide tube = 2 inches
$$
\n
$$
I = \pi \frac{(D_0^4 - D_i^4)}{64} = 3.2 inches^4
$$

for a **1 G** lateral force

$$
P = 25 \text{ pounds}
$$
  

$$
w = 0.4 \text{ pounds/inch}
$$

(structure is made of aluminum)

The total displacement if the sum of the displacement from the distributed load and the displacement from the concentrated load.

$$
\Delta X_{\text{total}} = \frac{w}{24 \text{ EI}} \quad (x^4 - 49^3 x + 39^4) \quad (\text{F.3})
$$
\n
$$
+ \frac{p}{6 \text{ EI}} \quad (29^3 - 39^3 x + x^3)
$$
\n
$$
X = \frac{0.4}{(24) (1 \times 10^7) (3.2)} \quad ((31)^4 + 4(48)^3 (31) + 3(48)^4)
$$
\n
$$
+ \frac{25}{6(1 \times 10^7) (3.2)} \quad (2(48)^3 - 3(48)^2 (31) + (31)^3)
$$

thus

 $\bar{z}$ 

$$
X = 0.00634 inches
$$

# Appendix **G**

## FORTRAN IV COMPUTER PROGRAM **USED** FOR

# APPROXIMATE **STACK** ANALYSIS

Dimensions of Stack

Height **= 150** feet

Outside Radius at the bottom of the stack <sup>=</sup>**7.21** feet Inside Radius at the bottom of the stack <sup>=</sup>*5.83* feet Outisde Radius at the top of the stack = *1.85* feet Inside Radius at the top of the stack - **1.25** feet

 $\sim$ 

```
c
      C
C
      PROGRAM TO CALCULATE THE SHEARING STRESSES ON THE STACK
      THE LOADING IS A COMBINATION OF A 100 MPH WIND
C
\mathbf{c}AND THE SHEAR USING THE SEADC DESIGN CODE
\mathbf cC.
      DIMENSION AH(26),ROH(26),PMASS(25),PMOM(25),FW(25),T(6),V(6)
      DIMENSION SHEAR(6,25)
932
      FORMAT(/2X.'OMEGA=({.597*3.1416)/L**12.0)*SQRT(E*I/M)'./2X.
     1 'SHEAR=1.5*K*C*W',/2X,'WHERE 1.5 IS A FACTOR INCREASING',
     2 ' SHEAR FOR CIRCULAR CROSS SECTION',/10X,'K=1.5 FOR BRITTLE',
     3 ' STRUCTURE',/10X,'W= WEIGHT OF STACK')
933
      FORMAT(/10X.'C=0.05/(PERIOD**0.333)',//2X,'A 190 MPH WIND ',
     1 'LOAD IS ALSO INCLUDED' )
\mathbf cC
      ROBOT = RADIUS AT THE BOTTOM OF THE STACK OUTSIDE
C
      ROTOP = RADIUS AT THE TOP OF THE STACK OUTSIDE
\mathbf cRIBOT = RADIUS AT THE BOTTOM OF THE STACK INSIDE
\mathbf{r}RITOP = RADIUS AT TOP OF THE STACK INSIDE
C.
      RITOP=2.5/2.0
      ROTOP=3.69/2.0
      RIBOT=11.67/2.0
      ROBOT=14.42/2.0
\mathbf{x}C.
      THE STACK IS ASSUMED TO BE BRITTLE
C.
      SHEAR = 1.5*K*C*W
\mathbf cSEACO DESIGN CODE GIVES
\mathbf cC.
\mathbf cC.
      THE STACK IS DIVIDED INTO 25 SECTIONS AND THE MASS OF EACH
\mathbf{C}SECTION IS DETERMINED
C
      THE MOMENT OF EACH SECTION ABOUT THE BASE IS DETERMINED
\mathbf cATOP=3.1416*(1.85**2.0-1.25**2.0)
      ABOT=3.1416*(7.21**2.0-5.835**2.0)
      ATAV=(ATOP+ABOT)/2.0
      TMASS=ATAV*150.0*125.0+400.0
      AH(1)=ATOP
      H = 0.0L = 1DO 1000 I=2,26
      HH= H+3.0
      HB=H+6.0RIH=RITOP+((RIBOT-RITOP)*HB/150.0)
      ROH(I)=ROTOP+((ROBOT-ROTOP)*HB/150.0)
      AH(I)=3.1416*(ROH(I)**2.0-RIH**2.0)
      AAVE=(AH(L)+AH(I))/2.0
      PMASS(L) =AAVE*6.0*125.0
      PMOM(L)=PMASS(L)*(150.0-HH)
      H=H+6.0L = L + 1IF(I-13) 1000,900,1000
900
      RIP=RIH*12.0
      ROP=ROH(I) *12.0
1000
     CONTINUE
      E = 20000000.0ſ.
C
      THE APPROXIMATE MOMENT OF INERTIA IS CALCULATED
C,
```
C

```
MOMI=(3.1416*((2.O*ROP)**4.0-(2.0*RIP)**4.0))/64.0
      ML=TMASS/(150f.o* 144.0)
      XL=(150.0*12.0)**2.0
      OMEGA= (0.597*3.1416)**2.O/XL)*SQRT(E*MOMI/ML)
      TAPP=1.O/OMEGA
      PMASS(1)=PMASS(1)+400.0
      PMnM(i)=PMOM(1)+(4f0.0*149.)
1001 F3RMAT(3X,'STACK SHEAR USING THE SEAOC DESIGN CODE')
l92 FIRMAT(/5X,'TOTAL STACK MASS = ',Fl0.2)
1005 F3RMAT(/3X,'FIRST OMEGA = ',FlC.5,4X,'PERIOD = ',F1'.5,2X,'SEC')
1003 FORMAT(/2X,'SECTION',3X,'CL HEIGHT',5X,'MASS',8X,'MOMENT',9X,
     1 'BOTTOM AREA')
      WRITE(6,1001)
      WRITE(6,932)
      WRITF(6.933)
      WRITF(6,1002) TMASS
      WRITE(6,1035) OMEGA,TAPP
      WRITE( 6,1003)
      H=147.0...DO 100 J=1,25
      L=J+1
1004 FORMAT(4X,14,6XF7.2,2XF12.',?X.F12.2,2X,F5.1)
      WRITE(6.1004) J.H.PMASS(J),PMOM(J),AH(L)
      H=H-6.0
100 CONTINUE
      TOTM=0.0
      DO 230 L=1,25
200 TITM=T3TM+PMOM(L)
C
C THE WIND LOADING IS APPLIED
      ROH(1 )=ROTOP
      DO 300 1=1,25
      l=1+1FW(I =((ROH(I)+ROH(L))*6.0*22.0)
300 CONTINUE
      A=1.0
      DO 400 1=1,6
      T(I)=TAPP/A
C
r THE SEACO DFSIGN CODE IS APPLIED TO OBTAIN THE SHEAR AT EACH
C SECTION
r
      V(I)=1.5*TMASS*0.05/(T(I)**0.333)
      A=A+1.0
400 CONTINUE
C
C THE SHEARS ARE SUMMED AND PRINTED OUT
      DO 5000 J=1,6
      SUMA=0.0
      SUMR=0.0
C
      DO 5000 1=1,25
      L = 1 + 1SUMB=SUMB+PMOM(I)
      R=SUMB/TOTM
      SUMA-SUMA+FW( I)
      SHEAR(JI)=((R*V(J)+SUMA)/(AH(L)*144.0))*1.5
5000 CONTINUE
     8000 FORMAT(//2X,'SECTTON',5X,'WIND SHEAR',5X,'TOTAL SHEAR IN PSI')
8l F3RMAT(/2X,'DYNAMIC SHEAR AT BASE =',F10.2,3X,'PERIOD=',FT.2,
     I 'SECONDS')
8002     FORMAT(5X.14.5X.F10.2.5X.F10.2)
C
C
C THE PROGRAM DETERMINES THE SHEARS FOR HIGHER FREQUENCIES AS
```
**C** WELL TO BE CONSERVATIVE **C D'3 700 J=1,6** WRTTF(6,8031) **V(J) ,T (J)** WRI **TE (6, 8000)** 0n **710 1=1,25** WRITF(6,802) **1,FW(I),SHEAR(J,I)** 700 CONTINUE **CALL** EXIT **END**  $\mathsf{C}$  $\mathbf c$ 

 $\sim 10$ 

 $\mathcal{A}^{\text{max}}_{\text{max}}$ 

 $\hat{\beta}$  and the following space  $\hat{\beta}$ 

#### Appendix H

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