Design of a Chain Flail Mower for Leveling Rough Terrain

by

Aaron Flores

Submitted to the Department of Mechanical Engineering in Partial Fulfillment of the Requirements for the Degree of

Bachelor of Science in Engineering as Recommended **by** the Department of Mechanical Engineering

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Accepted by:

Anette Hosoi Professor of Mechanical Engineering Undergraduate Officer

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Abstract

The flail mower is a piece of agricultural equipment that uses bladed attachments rotating around a drum to cut down bushes and grassy terrain. One major drawback to the flail mower is the rapid wear that happens almost immediately when any sort of rocks, gravel, or rough terrain are encountered. The goal of this project was to design and manufacture a prototype of a machine that could level rough terrain through repeating impacts as opposed to cutting with blades. The final design was a chain flail mower, a piece of equipment designed to level rough terrain that uses chains to repeatedly bludgeon rocks, gravel, and debris. The designed chain flail mower was assembled into an existing commercially available snow blower for testing.

Thesis Supervisor: Alexander H. Slocum Title: Pappalardo Professor of Mechanical Engineering

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1. Introduction

The goal of the project is to design a machine for leveling rough dirt terrain. The designed chain flail mower machine is modeled after a flail mower, which is designed for leveling grass-filled terrain. The flail mower (shown below) is a piece of agricultural equipment comprised of a rotating drum usually driven **by** a hydraulic motor attached to a tractor. Welded to the drum are numerous free-floating "Y" shaped blades with one degree of freedom usually attached **by** a bolt or pin. As the drum spins, the blades cut down grass at very high speeds (around **6000** fpm) and are also able to absorb impact through their bolted joint.

Figure 1: Image of a chain flail mower taken from http://scsc302.wikispaces.com/file/view/flail.jpg/187565895/flail.jpg

The drawbacks of the flail mower are that the blades at the ends of the flail wear almost immediately when they encounter rocks, dirt, or other sorts of rough terrain. Although these blades are replaceable, the quick wearing makes cutting rough terrain impractical. Thus, flail mowers are more effective for flat, grass-filled terrain and much less effective for uneven, rough, dirt-filled terrain.

Another source of inspiration for the chain flail mower is the mine flail pictured below. **A** mine flail is a piece of landmine clearing equipment that is either built into specialized vehicles or attached to the front of tanks **by** a boom. It is similar to the flail mower in that it has a rotating drum, but driven is **by** different types of motors ranging such as electrical, hydraulic, or combustion. The mine flail has numerous chains welded to the rotating drum and a large weight attached to the end of each chain. The chain and weight assembly rotates at high speeds and triggers and detonates mines. The key feature in the mine flail is the ability to handle rough terrain without the rapid wear of the flail mower because of the bludgeoning effect of the chains.

Figure 2: Image of the MineWolf Landmine clearing vehicle from MineWolf Systems. Taken from http://www.minewolf.com/products/medium-minewolf-mw330.html

2. Design

The primary functional requirement for the project was the ability to level rough, uneven terrain including rocks, gravel, and grass. Using the flail mower and the mine flail as guides, the chain flail mower uses chains attached to a cylindrical rod in order to create a bludgeoning effect for terrain and debris removal and leveling as opposed to the cutting effect from the blades of the flail mower. The chain flail mower uses four strips of steel welded on a steel cylindrical rod. The cylindrical rod has two shoulders for ball bearings, and is driven on the snow blower through a machine key at the end of the cylindrical rod. The chains are individually attached to holes drilled in the steel strips with a self-locking

shackle. The both sides of the rod have tapped holes drilled axially to constrain the rod within the machine, in this case the snow blower.

The design of the chain flail mower is different from the flail mower in that it is not a hollow drum, baut a solid rod because of the size difference in the equipment. The chain flail mower also features long welded strips as opposed to tabs. This allows the flail joints to still have a single degree of freedom, still remain in double shear so they don't have a bending moment, and be more cost-effective to manufacture because of less welds and less machining time. In order to address terrain leveling, the chains are placed as close together as possible with **0.100"** clearance between them. They are then staggered such that there is a sine-wave effect that pushes terrain to one side when the cylindrical rod rotates. The staggering can be changed such that terrain is pushed out, pushed in, or to one side. Because the chains are much closer together than the chains in the mine flail, the amount of terrain leveling is greater for the chain flail mower than the mine flail.

Figure 3: Self-locking shackle. Taken from http://cdn3.volusion.com/uybnd.syfpy/v/vspfiles/photos/WIC1202-2.jpg

Figure 4: Solidworks CAD model of the chain flail mower with one chain attached

3. Analysis

The mechanical design of the chain flail mower was driven **by** the theory that the lifecycle of a stressed system is approximately infinite when the system's material is steel and the stress it experiences is less than *50%* of its maximum yield stress', or that the factor of safety had to be greater than 2. Since the chain flail mower was designed to a commercially available snow blower, the stresses in the bearings and the key were assumed to be safely within the limits of failure.

3.1 Stress in the Chain and Shackle

The chains and shackles were modeled as a single point mass for an upper bound, first-order approximation of the stress. The equation describing a point mass in circular motion is given as:

$$
F_{net} = F_c = m \frac{v^2}{r}
$$
 (1)

Where F_{net} is the net force on the system, F_c is the centripetal force, m is the mass of each chain and shackle, *v* is the tangential speed, and *r* is the radius to the center of gravity along the length of the chain. The total mass of the chain and shackle assembly was used as the point mass for a worst-case scenario approach to the stress analysis. Substituting the equation 2 for angular velocity ω in revolutions per minute (rpm) and rearranging gives equation **3:**

$$
v = \omega \frac{2\pi}{60} r \tag{2}
$$

$$
F_c = m \frac{(2\pi\omega r)^2}{r \cdot 60^2} \tag{3}
$$

The stress was assumed to be linearly proportional to the force such that *50%* of the yield stress σ_y meant 50% of the maximum force shown in equation 2:

$$
F_c \sim \sigma_y \tag{4}
$$

What this shows is that the yield stress is directly proportional to the square of the rotational velocity of the cylindrical rod, such that as the drum rotates faster the stress will be higher.

3.2 Stress in the Welds

The welded strips were analyzed as having a fillet weld on both sides subjected to a shear force as shown below:

Figure *5:* **Diagram of a welded steel beam in bending using outside fillet welds ²**

Total stress τ_{total} on the weld joint was calculated² using equation 5:

$$
\tau_{total} = \sqrt{\tau_b + \tau_s} \tag{5}
$$

Where τ_b is the stress due to bending and τ_s is the stress due to shear. The bending and shear stresses were calculated² using equations 6, 7, 8.

$$
x = (d + 2h)^3 - d^3 \tag{6}
$$

$$
\tau_b = \frac{\text{s.5PA}(d+2h)}{bx} \tag{7}
$$

$$
\tau_s = \frac{0.71P}{bh} \tag{8}
$$

Where *P* is the load, **b** is the width of the beam, *d* is the depth of the beam, *h* is the thickness of the welds, and *A* is the area of the beam as shown in Fig. *5* above. The main disadvantage with this welded design over the flail mower is bending stress along the weak axis of the strips as opposed along the stiffest axis in the tabs on a conventional flail mower. This is mitigated **by** having a short bend radius and using a sufficient thickness for the strip.

4. Results

The original goal of the project was to develop a piece of agricultural equipment capable of leveling rough terrain including dirt and debris. **A** prototype chain flail motor was manufactured to fit the dimensions of a Toro snow blower³ and installed as shown below.

Figure 6: Detailed view of one end of the final manufactured chain flail mower

Figure 7: The designed chain flail mower prototype installed in a snow blower by replacing the original auger.

The manufactured chain flail mower shown is welded on both sides along the entire length of the strip. The shackles have a workload limit of 700lbs-f while the chain has a workload limit of 400lbs-f. Using equation **(1),** the maximum surface speed that the mower can use while still under the limiting workload limit is **6600** fpm.

Weld strength is more difficult to analyze since measured strengths are always less than theoretical due to external factors such as weld penetration and annealing. The yield stress for the welds on the chain flail mower was estimated to be much less than the **50%** yield criteria.

5. Conclusion

The chain flail mower uses chains with weights attached at the end of a chain in order to create a bludgeoning effect for terrain and debris removal and leveling as opposed to the cutting effect from the blades of the flail mower. The mower vehicle was manufactured and installed **by** replacing the auger of a snow blower with the finished design. The vehicle was not tested, and would need various safety precautions such as a safety shield to prevent injury from flying projectiles. Testing would need a way to

quantify the terrain leveling ability and comparing it to the leveling abilities of commercially available products flail mowers and mine flails. For commercial use, the chain flail mower would need to be scaled higher and would be powered through a hydraulic motor or an attachment to an existing drivetrain. Weld testing should be result in a reduction in the amount of weld beads needed for stich welds. Similarly, bolts or different shackles with vibration resistant fasteners should be used to withstand repeated impacts and would need thread-locking adhesive for maximum safety. Additionally, bushings would ideally be assembled in the joints connecting the chain to the rotating cylinder or drum to prevent wear on the joint. Last, more comprehensive analysis needs to be performed on the dynamics of repeated impacts on chains for a finer characterization of the yield stresses.

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